INTERNATIONAL **STANDARD**

ISO 8686-3

> First edition 1998-11-01

Cranes — Design principles for loads and load combinations —

Part 3:

Tower cranes

Appareils de levage à charge suspendue — Principes de calcul des charges et des combinaisons de charge es à tou es à tou vien circk to vien standards son conficient son

Partie 3: Grues à tour



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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through 1SO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 8686-3 was prepared by Technical Committee SO/TC 96, Cranes, Subcommittee SC 7, Tower cranes.

ISO 8686 consists of the following parts, under the general title Cranes Click to view the combinations:

- Part 1: General
- Part 2: Mobile cranes
- Part 3: Tower cranes
- Part 4: Jib cranes
- Part 5: Overhead travelling cranes and portal bridge cranes

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Cranes — Design principles for loads and load combinations —

Part 3:

Tower cranes

1 Scope

This part of ISO 8686 establishes the application of ISO 8686-1 for tower cranes, as defined in ISO 4306-3, and gives specific values for factors to be used.

2 Normative references

The following standards contain provisions which, through reference in this text, constitute provisions of this part of ISO 8686. At the time of the publication, the editions indicated were valid. All standards are subject to revision, and parties to agreements based on this part of ISO 8686 are encouraged to investigate the possibility of applying the most recent editions of the standards indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 4302:1981, Cranes — Wind load assessment.

ISO 4306-1:1990, Cranes — Vocabulary — Part 1: General.

ISO 4306-3:1991, Cranes — Vocabulary — Part 3: Tower cranes.

ISO 4310:1981, Cranes — Test code and procedures.

ISO 8686-1:1989, Cranes — Design principles for loads and load combinations — Part 1: General.

ISO 12485:—1), Cranes — Stability requirements of tower cranes.

3 Definitions

For the purposes of this part of ISO 8686, the definitions given in ISO 8686-1 apply.

4 Symbols and abbreviated terms

The symbols used are described in ISO 8686-1:1989, table 1.

¹⁾ To be published.

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5 General

The proof of competence of components shall be determined by either the limit state method or the allowable stress method as set out in ISO 8686-1.

Tower cranes which are subject to tipping and drifting shall be designed in accordance with ISO 12485.

6 Loads and applicable factors

Table 1 specifies factors Φ_n for dynamic effects which are used for load combinations listed in table 2, and also makes reference to ISO 8686-1 and other relevant International Standards.

The line numbers listed in the first column of table 1 are those shown in column 3 of table 2.

For those parts of the crane whose masses significantly decrease the resulting load effects and which have to be considered as "favourable", reduced partial load factors shall be applied.

Where the masses and their centres of gravity are determined by experiment (weighing), the factors in accordance with favourable effects 1.2.1 of table 2, line 1, shall be used.

Where the masses and their centres of gravity are calculated based on final piece lists, the factors in accordance with favourable effects 1.2.2 of table 2, line 1, shall be used.

Click to view the factors in accordance with favourable effects 1.2.2 of table 2, line 1, shall be used.

Table 1 — Factor Φ_n for dynamic effects

Line No. in table 2	Factor Φ_{n}	Clause references in ISO 8686-1	Guidance on values of factors $\Phi_{\it fl}$ and load factors References to other International Standards
1	Φ_1	6.1.1	$\Phi_1 = 1 \pm a$ $a = 0,1$
			$ \Phi_1 $ = 1 for design against tipping
2	Φ_2	6.1.2.2	Hoisting class HC 1 with a mininum value of 1,05
	Φ_3	6.1.2.3	
3	Φ_4	6.1.3.2	$arPhi_4$ = 1,1 is recommended for building site cranes — Other values may be used when the rail tracks tolerances (as agreed between user and manufacturer) vary from standard
			When using rigid body kinetic models:
4		6.1.4	Φ_5 = 1,2 if the acceleration or braking forces are changed with stepless control systems without backlash
and 5	Φ_5	and annex D	Φ_5 = 1,5 in other control systems where drive forces are acting on the crane practically free of backlash
3			Φ_{5} = 2 where considerable backlash exists
			Other values for $arPhi_5$ may be used when substantiated
6		6.1.5	Partial load factors shall be considered where appropriate
7		6.2.1.1	In-service wind in accordance with ISO 4302
8		6.2.1.2	Snow and ice loads need only be considered in special cases and then in accordance with regional conditions
9		6.2.1.3	Loads due to temperature variations need only be considered where appropriate according to regional and local conditions
10		6.2.2	Loads caused by skewing are negligible when using common undercarriages, otherwise requirements of ISO 8686-1:1989, annex F apply
11	Φ_2	6.1.2.2.2	Hoisting class HC 1
12		63.1	Regional out-of-service wind load-conditions in accordance with ISO 4302
13	Φ_6	6.3.2	Static test load = 1,25 × (Net load carried out in accordance with ISO 4310)
		R	Dynamic test load $\Phi_6 = 0.5 \times (1 + \Phi_2)$
14	Φ_{7}	6.3.3	Buffer forces need not be considered where the travelling velocity at contact with the buffer or end stop is less than 0,7 m/s
15		6.3.4	Tilting forces shall not be considered
16	Φ_5	6.3.5	Loads caused by emergency cut-out shall have a maximum value of the factor $\varPhi_{\rm 5}$ of 2,0
17		6.3.6	Load caused by failure of mechanism or components shall be considered where appropriate
18		6.3.7	Excitation effects shall be considered where appropriate

Table 2 — Loads and load combinations — Tower cranes

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		Ś			Loa	Load combinations	binat	ions A	_	Loa	8 9	틽	Load combinations	8	\dashv	-	<u> </u>	oad	gmo	Load combinations	Suc	-	-	Т
Categories of loads		List of the loads t_i	ds f	Line No.	Partial load	E	A 2	¥3	<u>₹</u>	Partial load	19	B	 B3		B5 - R	Partial load		8		<u>2</u>		C6 C7		8
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	acceleration,	2. Mass of the gross load	oss load	7	1,34	ϕ_2	$\Phi_{_{3}}$	-		1,22	ϕ_2	Φ ^s	-			-,-		u		-	•	_		-
Regular	impacts	3. Masses of crane and hoist load travelling on uneven surface	ne and hoist load neven surface	8	1,22				Φ_4	1,16				Φ_4	Φ_4							`		
(see 6.1)	Acceleration	4. Masses of	4.1 Hoist drives excludes	4	Silv	ф 2	ϕ_5				ϕ_5	ϕ_5							ϕ_5					
	From drives	gross load	4.2 Hoist drives includes	S	1,34	*O		φ	φ ²	1,22			Φ_5	Φ_{5}		<u>+,</u> -								
	Displacements	5. See 6.1.5		9	1,16	1	1/2	1,0	1	1,1	-	+	1	1	-	1,05	1	-	-	-		-		1
		1. In-service wind loads	loads	4				12		1,16	-	-	-	-	-						\vdash	\vdash	\vdash	
Occasional	Effects of	2. Snow and ice loads	oads	8					~	1,22	1	-	1	1	-	1,1		1						
(see 6.2)	Omina	3. Temperature variations	ariations	6					X	1,16	-	-	1	-	-	1,05		-						
	Skewing	4. See 6.2.2		10					r 	1,16					1									
	1. Hoisting a grounded load	rounded load		11							X					1,1	Φ_2							
	2. Out-of service wind loads	se wind loads		12												1,1		-						
	3. Test loads			13									C			1,1			$\phi_{\rm e}$					
Exceptional	Exceptional 4. Buffer forces	6		14									Q			1,1				ϕ_{7}				
(see 6.3)	5. Tilting forces			15										20		1,1								
	6. Emergency cut-out	cut-out		16										b	\C	1,1					5	$\phi_{\rm s}$		
	7. Failure of mechanism	echanism		17											ج	1,1								
	8. Vibration of	8. Vibration of the crane's foundation	ion	18					\dashv					\dashv		0		\neg						-
	Safety factor fo	Safety factor for calculating allowable stresses	able stresses $\gamma_{ m f}$	19			1,48	48					1,34			200	C-			1,22	<u></u>			
	Resistence coefficient $~\gamma_{ m m}$	əfficient γ_{m}		20	1,1					1,1					\vdash	1,1								