

NFPA 8506

Standard on Heat Recovery Steam Generator Systems

1998 Edition



National Fire Protection Association, 1 Batterymarch Park, PO Box 9101, Quincy, MA 02269-9101
An International Codes and Standards Organization

Copyright ©
National Fire Protection Association, Inc.
One Batterymarch Park
Quincy, Massachusetts 02269

IMPORTANT NOTICE ABOUT THIS DOCUMENT

NFPA codes and standards, of which the document contained herein is one, are developed through a consensus standards development process approved by the American National Standards Institute. This process brings together volunteers representing varied viewpoints and interests to achieve consensus on fire and other safety issues. While the NFPA administers the process and establishes rules to promote fairness in the development of consensus, it does not independently test, evaluate, or verify the accuracy of any information or the soundness of any judgments contained in its codes and standards.

The NFPA disclaims liability for any personal injury, property or other damages of any nature whatsoever, whether special, indirect, consequential or compensatory, directly or indirectly resulting from the publication, use of, or reliance on this document. The NFPA also makes no guaranty or warranty as to the accuracy or completeness of any information published herein.

In issuing and making this document available, the NFPA is not undertaking to render professional or other services for or on behalf of any person or entity. Nor is the NFPA undertaking to perform any duty owed by any person or entity to someone else. Anyone using this document should rely on his or her own independent judgment or, as appropriate, seek the advice of a competent professional in determining the exercise of reasonable care in any given circumstances.

The NFPA has no power, nor does it undertake, to police or enforce compliance with the contents of this document. Nor does the NFPA list, certify, test or inspect products, designs, or installations for compliance with this document. Any certification or other statement of compliance with the requirements of this document shall not be attributable to the NFPA and is solely the responsibility of the certifier or maker of the statement.

NOTICES

All questions or other communications relating to this document and all requests for information on NFPA procedures governing its codes and standards development process, including information on the procedures for requesting Formal Interpretations, for proposing Tentative Interim Amendments, and for proposing revisions to NFPA documents during regular revision cycles, should be sent to NFPA headquarters, addressed to the attention of the Secretary, Standards Council, National Fire Protection Association, 1 Batterymarch Park, P.O. Box 9101, Quincy, MA 02269-9101.

Users of this document should be aware that this document may be amended from time to time through the issuance of Tentative Interim Amendments, and that an official NFPA document at any point in time consists of the current edition of the document together with any Tentative Interim Amendments then in effect. In order to determine whether this document is the current edition and whether it has been amended through the issuance of Tentative Interim Amendments, consult appropriate NFPA publications such as the *National Fire Codes*® Subscription Service, visit the NFPA website at www.nfpa.org, or contact the NFPA at the address listed above.

A statement, written or oral, that is not processed in accordance with Section 16 of the Regulations Governing Committee Projects shall not be considered the official position of NFPA or any of its Committees and shall not be considered to be, nor be relied upon as, a Formal Interpretation.

The NFPA does not take any position with respect to the validity of any patent rights asserted in connection with any items which are mentioned in or are the subject of this document, and the NFPA disclaims liability of the infringement of any patent resulting from the use of or reliance on this document. Users of this document are expressly advised that determination of the validity of any such patent rights, and the risk of infringement of such rights, is entirely their own responsibility.

Users of this document should consult applicable federal, state, and local laws and regulations. NFPA does not, by the publication of this document, intend to urge action that is not in compliance with applicable laws, and this document may not be construed as doing so.

Licensing Policy

This document is copyrighted by the National Fire Protection Association (NFPA). By making this document available for use and adoption by public authorities and others, the NFPA does not waive any rights in copyright to this document.

1. Adoption by Reference – Public authorities and others are urged to reference this document in laws, ordinances, regulations, administrative orders, or similar instruments. Any deletions, additions, and changes desired by the adopting authority must be noted separately. Those using this method are requested to notify the NFPA (Attention: Secretary, Standards Council) in writing of such use. The term “adoption by reference” means the citing of title and publishing information only.

2. Adoption by Transcription – **A.** Public authorities with lawmaking or rule-making powers only, upon written notice to the NFPA (Attention: Secretary, Standards Council), will be granted a royalty-free license to print and republish this document in whole or in part, with changes and additions, if any, noted separately, in laws, ordinances, regulations, administrative orders, or similar instruments having the force of law, provided that: (1) due notice of NFPA’s copyright is contained in each law and in each copy thereof; and (2) that such printing and republication is limited to numbers sufficient to satisfy the jurisdiction’s lawmaking or rule-making process. **B.** Once this NFPA Code or Standard has been adopted into law, all printings of this document by public authorities with lawmaking or rule-making powers or any other persons desiring to reproduce this document or its contents as adopted by the jurisdiction in whole or in part, in any form, upon written request to NFPA (Attention: Secretary, Standards Council), will be granted a nonexclusive license to print, republish, and vend this document in whole or in part, with changes and additions, if any, noted separately, provided that due notice of NFPA’s copyright is contained in each copy. Such license shall be granted only upon agreement to pay NFPA a royalty. This royalty is required to provide funds for the research and development necessary to continue the work of NFPA and its volunteers in continually updating and revising NFPA standards. Under certain circumstances, public authorities with lawmaking or rule-making powers may apply for and may receive a special royalty where the public interest will be served thereby.

3. Scope of License Grant – The terms and conditions set forth above do not extend to the index of this document.

(For further explanation, see the Policy Concerning the Adoption, Printing, and Publication of NFPA Documents, which is available upon request from the NFPA.)

Copyright © 1998 NFPA, All Rights Reserved

NFPA 8506

Standard on

Heat Recovery Steam Generator Systems

1998 Edition

This edition of NFPA 8506, *Standard on Heat Recovery Steam Generator Systems*, was prepared by the Technical Committee on Heat Recovery Steam Generators, released by the Technical Correlating Committee on Boiler Combustion System Hazards, and acted on by the National Fire Protection Association, Inc., at its Annual Meeting held May 18–21, 1998, in Cincinnati, OH. It was issued by the Standards Council on July 16, 1998, with an effective date of August 5, 1998, and supersedes all previous editions.

Changes other than editorial are indicated by a vertical rule in the margin of the pages on which they appear. These lines are included as an aid to the user in identifying changes from the previous edition.

This edition of NFPA 8506 was approved as an American National Standard on August 6, 1998.

Origin and Development of NFPA 8506

With the increased use of heat recovery steam generators (HRSGs) in industry, a technical committee was formed in 1993 to prepare a standard covering heat recovery steam generators. This document is the result of the work of this committee. This is the second edition of NFPA 8506 and is similar in organization to the other documents in the 8500 series boiler combustion system hazards standards.

The 1998 edition has incorporated several major changes. Most notable are the revision of purge requirements, the addition of mandatory language for combustion turbine exhaust bypass systems for unfired HRSGs, the elimination of nonmandatory language, and the addition of new definitions.

Technical Correlating Committee on Boiler Combustion System Hazards

Dale E. Dressel, *Chair*
Solutia Inc., MO [U]

Merton W. Bunker, Jr., *Nonvoting Secretary*
Nat'l Fire Protection Assn., MA

Courtney D. Alvey, Baltimore, MD [SE]
Terry Michael Bartels, Kansas City Power & Light Co., MO [U]
Johnny W. (Bill) Bass, Forney Corp., TX [M]
James R. Bostick, Bailey Controls Co., OH [M]
Thaddeus Bukowski, Underwriters Laboratories Inc., IL [RT]
William E. Cunningham, Jr., Raytheon Engr & Constructors, MA [SE]
John C. deRuyter, The DuPont Company, DE [U]
Robert S. Elek, Kemper Insurance, OH [I]
Ronald E. Fringeli, J&H Marsh & McLennan, OH [I]
Steven K. Funk, The DuPont Co., TX [U] Rep. American Petroleum Inst.
Gordon G. Gaetke, Union Carbide Corp., WV [U]

Masaaki Kinoshita, Mitsubishi Heavy Industries Ltd, Japan [M]
Donald J. L. Lin, Qilin Inc., TX [SE]
Francis X. Maskol, Honeywell Inc., GA [M]
Russell N. Mosher, American Boiler Mfrs. Assn., VA [M]
Jerry J. Moskal, ABB Combustion Engr, Inc., CT [M]
Michael C. Polagye, Factory Mutual Research Corp., MA [I]
Nancy C. Polosky, Babcock & Wilcox, OH [M]
George P. Seroka, The Detroit Edison Co., MI [U]
James L. Sherman, Baltimore Gas & Electric Co., MD [U]
Peter J. Gore Willse, Industrial Risk Insurers, CT [I]
Henry K. Wong, Foster Wheeler Energy Corp., NJ [M]

Alternates

Tetsuya Fujino, Mitsubishi Heavy Industries America, Inc., CA [M]
(Alt. to M. Kinoshita)
Raymond J. Heitland, Solutia, Inc., FL [U]
(Alt. to D. E. Dressel)
Dennis P. Jenkins, Kemper Nat'l Insurance Cos., NC [I]
(Alt. to R. S. Elek)
Gerald E. McCullion, Honeywell Inc., GA [M]
(Alt. to F. X. Maskol)
John P. O'Rourke, ABB Combustion Engr, Inc., CT [M]
(Alt. to J. J. Moskal)

Richard J. Wachter, Industrial Risk Insurers, CT [I]
(Vot. Alt. to IRI Rep.)
J. C. Waung, Babcock & Wilcox Co., OH [M]
(Alt. to N. C. Polosky)
Harold R. Yates, The Detroit Edison Co., MI [U]
(Alt. to G. P. Seroka)
William G. Yeich, Exxon Research and Engr Co., NJ [U]
(Alt. to S. K. Funk)

Nonvoting

William H. Axtman, Gray Gull Assoc., Inc., VA
Shelton Ehrlich, Palo Alto, CA
Thomas B. Hamilton, Hamilton Consulting Services, NC

Robert P. Kaltenbach, Burns & McDonnell Engr, MO
Peter B. Matthews, Hartford Steam Boiler Insp & Ins Co., CT

Merton W. Bunker, Jr., NFPA Staff Liaison

This list represents the membership at the time the Committee was balloted on the text of this edition. Since that time, changes in the membership may have occurred. A key to classifications is found at the back of this document.

NOTE: Membership on a committee shall not in and of itself constitute an endorsement of the Association or any document developed by the committee on which the member serves.

Committee Scope: This Committee shall have primary responsibility for documents on the reduction of combustion system hazards in single- and multiple-burner boilers with a heat input rate of 12,500,000 Btu/hr and above. This includes all fuels. This Committee also is responsible for documents on the reduction of hazards in pulverized fuel systems, fluidized-bed boilers, heat recovery steam generators, and stoker-fired boilers, at any heat input rate.

Technical Committee on Heat Recovery Steam Generators

Peter B. Matthews, Chair

Hartford Steam Boiler Insp & Ins Co., CT [I]

Samuel H. Allen, The Dow Chemical Co., U.S.A., LA [U]

Thomas J. Brown, Air Products & Chemicals, Inc., PA [U]

David Paul Cannon, Houston Industry Power, TX [U]

Ian M. Clark, ABB Combustion Engr Services, Inc., CT [M]

Ralph S. Clemens, InterGen, MA [U]

John H. Conroy, Forney Corp., TX [M]

James W. Dunlap, DuPont Power, DE [U]

Gordon G. Gaetke, Union Carbide Corp., WV [U]

Kris A. Gamble, Black & Veatch, KS [SE]

Paul A. Giansiracusa, United Engr & Constructors, MA [SE]

Alfred W. Grusi, John Zink Co./Division of Koch Engr, Co., OK [M]

Harold F. Honath, Jr., Coen Co., Inc., CA [M]

Robert P. Kaltenbach, Burns & McDonnell Engr, MO [SE]

George Y. Keller, Burns & Roe Enterprises, Inc., NJ [SE]

Steven J. Knott, Westinghouse Electric Corp., FL [M]

Steven A. Lefton, Aptech Engr Services, CA [SE]

Gerald E. McCullion, Honeywell Inc., GA [M]

Russell N. Mosher, American Boiler Mfrs. Assn., VA [M]

Martin C. Nygard, Nooter/Eriksen, MO [M]

Timothy R. Parrott, Travelers Insurance Co., IL [I]

Rep. American Insurance Services Group, Inc.

Owen M. Preston, Factory Mutual Engr, FL [I]

John Van Name, Raytheon Engr & Constructors, NY [SE]

Joseph M. Vavrek, Sargent & Lundy, IL [SE]

Alternates

Donald W. Bairley, ABB Combustion Engr Systems, Inc., CT [M]

(Alt. to I. M. Clark)

Rick Fiorenza, Coen Co., Inc., CA [M]

(Alt. to H. F. Honath, Jr.)

Allan G. Gilson, Burns & McDonnell Engr, MO [SE]

(Alt. to R. P. Kaltenbach)

W. J. Johnson, The Dow Chemical Co., TX [U]

(Alt. to S. H. Allen)

Francis X. Maskol, Honeywell Inc., GA [M]

(Alt. to G. E. McCullion)

William Method, The Pritchard Corp., KS [SE]

(Alt. to K. A. Gamble)

Michael C. Polagye, Factory Mutual Research Corp., MA [I]

(Alt. to O. M. Preston)

Stephen L. Somers, John Zink Co./Division of Koch Engr, Co., OK [M]

(Alt. to A. W. Grusi)

Stephen B. Statz, Raytheon Engr & Constructors, NY [SE]

(Alt. to J. Van Name)

Robert H. Warburton, Jr., J. Makowski Mgmt., Co., RI [U]

(Alt. to R. S. Clemens)

Merton W. Bunker, Jr., NFPA Staff Liaison

This list represents the membership at the time the Committee was balloted on the text of this edition. Since that time, changes in the membership may have occurred. A key to classifications is found at the back of this document.

NOTE: Membership on a committee shall not in and of itself constitute an endorsement of the Association or any document developed by the committee on which the member serves.

Committee Scope: This Committee shall have primary responsibility for documents covering the operation of heat recovery steam generators and the related reduction of combustion system hazards and prevention of boiler furnace explosions. This includes all fuels at any heat input rate.

Contents

Foreword	8506-5	6-6 Operator Interface	8506-18
Chapter 1 Introduction	8506-5	Chapter 7 Purge, Startup, Operation, and Shutdown of HRSG Systems.	8506-18
1-1 Scope.	8506-5	7-1 General.	8506-18
1-2 Purpose.	8506-5	7-2 General Operating Requirements	8506-19
Chapter 2 General	8506-6	7-3 Cold Start Preparation.	8506-19
2-1 HRSG Fires and Explosions	8506-6	7-4 Combustion Turbine Purge and Light-Off	8506-19
2-2 HRSG Implosions	8506-6	7-5 Duct Burner Purge and Light-Off	8506-20
2-3 Manufacture, Design, and Engineering.	8506-6	7-6 Normal Operation	8506-21
2-4 Basic Operating Objectives.	8506-7	7-7 Normal Shutdown.	8506-21
Chapter 3 Definitions	8506-7	7-8 Normal Hot Restart	8506-21
3-1 Definitions	8506-7	7-9 Duct Burner Emergency Shutdown. . . .	8506-21
Chapter 4 Project Coordination	8506-10	Chapter 8 Inspection, Maintenance, Training, and Safety	8506-22
4-1 General.	8506-10	8-1 Equipment Inspection and Maintenance	8506-22
4-2 Project Inception	8506-10	8-2 Training	8506-22
4-3 Design	8506-11	Chapter 9 Combustion Turbine Exhaust Bypass Systems for Unfired HRSGs.	8506-23
4-4 Construction and Installation	8506-11	9-1 General.	8506-23
4-5 Initial Training	8506-11	9-2 Purge.	8506-23
4-6 Commissioning	8506-11	9-3 Damper Requirements.	8506-23
Chapter 5 Equipment	8506-11	Chapter 10 Fully Fired Systems	8506-24
5-1 General.	8506-11	Chapter 11 Referenced Publications	8506-24
5-2 Combustion Turbine	8506-11	Appendix A Explanatory Material	8506-24
5-3 HRSG Fuel-Burning System	8506-12	Appendix B Industry Experience.	8506-34
5-4 HRSG Enclosure	8506-14	Appendix C Gas Bypass and Alternate Burner Systems.	8506-34
5-5 Selective Catalytic Reduction.	8506-15	Appendix D Referenced Publications.	8506-36
5-6 Electrical	8506-15	Appendix E Additional References.	8506-37
Chapter 6 Controls, Monitoring, Alarms, and Interlocks	8506-15	Index	8506-37
6-1 Control Functions.	8506-15		
6-2 Monitoring	8506-15		
6-3 Alarms	8506-16		
6-4 Interlocks.	8506-16		
6-5 Burner Management System Logic	8506-18		

NFPA 8506**Standard on****Heat Recovery Steam Generator Systems****1998 Edition**

NOTICE: An asterisk (*) following the number or letter designating a paragraph indicates that explanatory material on the paragraph can be found in Appendix A.

Information on referenced publications can be found in Chapter 11 and Appendix D.

FOREWORD

This document is intended to be used in the design, installation, operation, and maintenance of heat recovery steam generator (HRSG) systems. It is not possible for this standard to encompass the specific hardware applications, nor should it be considered a “cookbook” for the design of a safe HRSG system.

The simplest combined cycle plant automatically has certain hazards that are common to all designs. Coupling various designs of heat recovery units with combustion turbines of varying characteristics in different configurations (such as varying damper arrangements) can produce very unique hazards.

The potential ineffective use of the combustion turbine as the source of the purge and potential sources of substantial fuel entering the HRSG from normal and false starts are major considerations that need to be addressed.

Other concerns include special provisions such as automatic transfer during transients, multiple stacks that can create reverse flows, internal maintenance of the HRSG with the combustion turbine in operation, multiplicity of cross connections between units to prevent shutdown, and/or fitting it into a small space using finned tubes which are more sensitive to temperature and subject to iron fires.

Insufficient failure analysis of arrangements, configurations, and equipment can increase the number of damaging incidents, lost production, and the possibility of personal injury or death.

It is vital that the designer of the combustion turbine and any burner safety system(s) be completely familiar with the features, characteristics, and limitations of the specific hardware and also possess a thorough understanding of this standard and its intent.

Chapter 1 Introduction**1-1 Scope.**

1-1.1 This standard shall apply to steam generators that recover heat from a combustion turbine. Purge requirements shall apply to all fired and unfired heat recovery steam generator (HRSG) systems regardless of the fuel(s) used.

1-1.1.1 The combustion turbine shall be permitted to fire the following fuels either alone or in combination:

- (a) Natural gas
- (b) No. 2 fuel oil
- (c) Kerosene
- (d) JP4 jet fuel
- (e) Diesel fuel

1-1.1.2 The HRSG shall be permitted to be unfired, or supplemental firing shall be permitted to be provided from natural gas or fuel oil, as defined in this standard.

1-1.2 The intent of this standard shall not be construed to dictate the methods or details of the combustion turbine manufacturer’s product or control system. Specific functional considerations are identified for proper interfacing related to the safety aspects of the combined combustion turbine and the HRSG.

1-1.3 This standard shall not specifically address the firing of fuels other than those specified in 1-1.1.1 and 1-1.1.2. This standard shall apply to other fuels not covered by 1-1.1.1 and 1-1.1.2 (*see 1-2.2*). Other fuels have unique characteristics and shall require additional design and operation considerations for fuel handling and burning.

1-1.4 This standard shall not cover simultaneous firing of more than one fuel in the supplemental firing of the steam generator. Simultaneous firing shall involve requirements that are not addressed by this document.

1-1.5 This standard is not retroactive. This standard shall apply to new installations and to major alterations or extensions of existing equipment for the preparation and burning of fuel contracted for 6 months subsequent to the date of issue of this document.

1-1.6 Since the standard is based on the present state of the art, its application to existing installations shall not be required. Nevertheless, operating companies are encouraged to adopt those features of the standard that are considered applicable and reasonable for existing installations.

1-1.7 This standard shall not address multiple combustion turbines exhausting into a single HRSG, nor does it address single or multiple combustion turbines exhausting into multiple HRSGs through a header. Any system applying this concept shall require special design considerations that are not addressed in this standard.

1-1.8 The standard shall be applied with consideration of the document as a whole. Chapters 1 and 2 are introductory and provide general guidelines and focus. Where any real or perceived conflict exists, the guidelines outlined in subsequent chapters shall be followed.

1-2 Purpose.

1-2.1 The purpose of this standard shall be as follows:

- (a) To contribute to operating safety
- (b) To prevent explosions, implosions, and uncontrolled fires in HRSG sections and exhaust ductwork
- (c) To establish minimum standards for the design, installation, operation, and maintenance of heat recovery steam generators and their fuel-burning, air supply, and combustion products removal systems
- (d) To require the coordination of operating procedures and components, control systems, interlocks, and structural design
- (e) To require the establishment of training programs in equipment operation and maintenance, for both new and existing personnel, to ensure that minimum standards for operation and maintenance are understood and followed

1-2.2 No standard can guarantee the elimination of HRSG explosions and implosions. Technology in this area is under constant development and is reflected in revisions to this stan-

dard. The user of this standard shall recognize the complexity of firing with regard to the type of equipment used and the characteristics of the fuel. Therefore, the designer is cautioned that the standard is not a design handbook. The standard does not eliminate the need for the engineer or for competent engineering judgment. It is intended that a designer capable of applying more complete and rigorous analysis to special or unusual problems shall have latitude in the development of such designs. In such cases, the designer shall be responsible for demonstrating the validity of the approach.

1-2.3 Emphasis shall be placed on the importance of component coordination and on knowledge of expected operating conditions, along with adequate structure strength, proper operation and maintenance procedures, combustion and draft control equipment, safety interlocks, alarms, trips, and other related controls that are essential to proper operation.

1-2.4 The effect of gas cleanup systems located within or downstream of the HRSG shall be considered in the design and operation of the system.

Chapter 2 General

2-1 HRSG Fires and Explosions.

2-1.1 The basic cause of uncontrolled fires or explosions in an HRSG system is the ignition of an accumulated combustible mixture within the HRSG enclosure.

2-1.2 A dangerous combustible mixture within the HRSG sections and ductwork consists of the accumulation of combustibles mixed with air in proportions that result in uncontrolled combustion when an ignition source is supplied. An explosion might result from ignition of this accumulation if the quantity of combustible mixture and the proportion of air to fuel are such that an explosive force is created within the HRSG enclosure. The magnitude and intensity of the explosion depends on both the relative quantity of combustibles that has accumulated and the proportion of air that mixes with them at the moment of ignition. Explosions, including "puffs," are the result of improper procedures used by operating personnel, improper design of equipment or control systems, or equipment or control system malfunction.

2-1.3 Numerous situations that produce uncontrolled combustion conditions can arise in connection with the operation of an HRSG. These situations include the following:

- (a) An interruption of the fuel or combustion air supply or ignition energy to the burners, sufficient to result in a momentary loss of flame, followed by the delayed reignition of an accumulation of combustibles
- (b) Fuel leakage into the enclosure and the ignition of the accumulation by a spark or other source of ignition
- (c) Repeated unsuccessful attempts to light off the combustion turbine or supplementary fired burners without appropriate purging, resulting in the accumulation of an explosive mixture
- (d) The accumulation of an explosive mixture of fuel and combustion air as a result of loss of flame or incomplete combustion at one or more burners in the presence of other burners operating normally or during light-off of additional burners
- (e) The accumulation of an explosive mixture of fuel and combustion air as a result of a complete burner flameout

and the ignition of the accumulation by a spark or other ignition source, such as an attempt to light off burners without adequate purge

2-1.4 An examination of reports of steam generator explosions suggests that occurrences of small explosions, puffs, or near-misses are far more frequent than is usually recognized. Improved instrumentation, safety interlocks and protective devices, proper operating sequences, and a clearer understanding of the problem by designers, operators, and maintenance personnel can greatly reduce the risks and actual incidence of HRSG explosions.

2-1.5 In an HRSG with supplementary firing, upset conditions or control malfunctions can lead to an air/fuel mixture that could result in a flameout followed by reignition after a combustible mixture has been reestablished. Dead pockets might exist in HRSG sections and ductwork or other parts of the unit where combustible mixtures can accumulate under upset conditions. These accumulations can ignite with explosive force in the presence of an ignition source.

2-1.6 Statistics indicate that human error is a contributing factor in the majority of explosions. It is important to consider whether the error is the result of any of the following conditions:

- (a) Unfavorable operating characteristics of the equipment or its controls
- (b) Lack of functional coordination of the various components of the steam-generating system and its controls
- (c) Lack of understanding of, or failure to follow, proper operating procedures, safeguards, and equipment operation recommendations

2-1.7* HRSG Enclosure, Fin-Metal Tube Fires.

CAUTION

Due to misoperation, there have been occurrences of fin-metal tube fires in HRSGs, and there is a need to address this potential. See Appendix E for specific articles related to iron fires which in turn have references to previous literature.

2-1.7.1 Consideration shall be given to detecting and alarming fin-metal tube fires.

2-1.7.2 A procedure shall be provided to contain (i.e., confine or seal off) a fin-metal tube fire and to protect the buildings and adjacent equipment.

2-1.7.3 The initial response shall include the removal of all fuel from the combustion turbine and the HRSG.

2-2 HRSG Implosions. For HRSGs that use an induced draft fan where the potential exists for excessively low HRSG enclosure gas side pressures, the HRSG enclosure shall conform with the requirements of NFPA 8502, *Standard for the Prevention of Furnace Explosions/Implosions in Multiple Burner Boilers*, Chapter 5.

2-3 Manufacture, Design, and Engineering.

2-3.1 The owner or the owner's agent shall, in cooperation with the manufacturer, ensure that the unit is not deficient in apparatus that is necessary for proper operation with respect to pressure parts, fuel-burning equipment, combustion air and fuel, and safe light-off and maintenance of stable flame.

2-3.2 All fuel systems shall include provisions to prevent foreign substances from interfering with the fuel supply to the duct burner.

2-3.3 An evaluation shall be made to determine the optimum integration of manual and automatic safety features considering the advantages and disadvantages of each trip function.

CAUTION

The maximum number of automatic trip features does not necessarily provide for maximum overall safety. Some trip actions result in additional operations that increase exposure to hazards.

2-3.4 This standard shall require a minimum degree of automation. However, in the cases of more complex plants, plants with increased automation, and plants designed for remote operation, the following minimum provisions shall be required:

- (a) Information on significant operating events that allow the operator to make a rapid evaluation of the operating situation
- (b) Continuous and usable displays of variables in the normal control location that allow the operator to avoid unsafe operation
- (c) Additional automated control points that allow a remotely located operator to control the unit safely
- (d) In-service maintenance and checking of system functions without impairment of the reliability of the overall control system
- (e) Control areas environmentally designed to provide suitable conditions for personnel, control equipment, or both
- (f) Visual displays and alarms that are easily identifiable and readable

2-3.5 The duct burner piping and equipment shall be designed and constructed to prevent the formation of hazardous concentrations of combustible gases under any operating conditions.

2-4 Basic Operating Objectives.

2-4.1 Basic operating objectives shall include the requirements of this section.

2-4.1.1 Standard operating procedures shall be established that result in well-defined and controlled operations. All operating procedures shall be rigidly enforced at all times.

2-4.1.2 Interlocks shall be used to minimize improper operating sequences and to interrupt sequences when conditions are unsafe for continuation.

2-4.1.3 No interlocks shall be bypassed during start-up or operation of the unit unless the bypass is alarmed and is governed by operating procedures.

2-4.1.4 The mass flow of combustion air to the duct burner system shall be maintained at or above its purge rate and within the duct burner design operating range during all operations of the duct burner system.

2-4.2 Written operating procedures and detailed checklists for operator guidance shall be provided for achieving these basic operating objectives. All manual and automatic functions shall be included in these procedures and checklists.

Chapter 3 Definitions

3-1 Definitions. The following definitions shall apply to this standard.

Air, Combustion. The air used to fuel the combustion process. For duct burners, this generally is combustion turbine exhaust. (See *Air, Primary* and *Air, Vitiated*.)

Air, Excess. Air supplied for combustion in excess of theoretical air. This is not the same as *Air-Rich* as defined in this section.

Air, Furnace Purge. See Purge, Combustion Turbine; Purge, Duct Burner; and Purge Rate.

Air, Primary. The air that is contained in the combustion turbine exhaust.

Air, Seal. Air supplied to any device at pressure for the specified purpose of minimizing contamination.

Air, Secondary. The air for combustion supplied to the burners by the forced draft fan.

Air, Stoichiometric. See Air, Theoretical.

Air, Tertiary. The air supplied to certain types of burners to cool the burner metal or to improve the combustion process.

Air, Theoretical. The chemically correct quantity of air needed for complete combustion of a given quantity of a specific fuel.

Air, Vitiated. Air with an oxygen content below normal, for example, combustion turbine exhaust.

Air/Fuel Ratio. A ratio of air to fuel supplied to a combustion process.

Air-Rich. Indicates a ratio of air to fuel supplied to a combustion process that provides more than the minimum excess air needed for optimum combustion of the fuel.

Alarm. An audible or visible signal indicating an off-standard or abnormal condition.

Approved.* Acceptable to the authority having jurisdiction.

Atomizer. A device in a burner that emits liquid fuel in a finely divided state.

Atomizer, Mechanical. A device in an oil burner that emits liquid fuel in a finely divided state without using an atomizing medium.

Atomizing Medium. A supplementary fluid, such as steam or air, that assists in breaking down liquid fuel into a finely divided state.

Augmented Air Firing. Supplementary firing with the addition of air at the duct burners to support and stabilize combustion or to improve emissions.

Authority Having Jurisdiction.* The organization, office, or individual responsible for approving equipment, an installation, or a procedure.

Auxiliary Firing. See Supplementary Firing.

Availability. The percent of time during a fixed period that a generating unit is capable of providing functional operating service.

Burner. A device or group of devices for the introduction of fuel and air at the velocities, turbulence, and concentration necessary to maintain ignition and combustion of the fuel.

Burner Management System. The control system dedicated to combustion safety, operator assistance in the starting and stopping of fuel-burning equipment, and prevention of misoperation and damage. The burner management system includes the following functions specified in this standard:

- (a) Interlock system
- (b) Fuel trip system

- (c) Master fuel trip system
- (d) Master fuel trip relay
- (e) Flame monitoring and tripping systems
- (f) Ignition subsystem
- (g) Duct burner subsystem

Combustion Control System. The control system that regulates the fuel input (and air, if applicable) to maintain continuous combustion and stable flame. This control system might include drum level, desuperheater spray, or draft control where applicable.

Combustion Turbine. A turbine in which the rotating element is actuated by the pressure of combustion gases on curved vanes.

Commercial Operation. The date that the full plant capacity is formally added to the power grid.

Commissioning. The time period of plant testing and operation between initial operation and commercial operation.

Damper. A mechanical device for controlling or blocking gas flow to a specific component.

Damper, Air Inlet.* A damper used to admit fresh air into the HRSG, either as an alternative method of purging or to supply combustion air when the combustion turbine is out of service.

Damper, Bypass.* A damper that allows combustion turbine exhaust gases to bypass the HRSG.

Damper, Diverter. A damper that is used to select alternate gas paths.

Damper, HRSG Isolation.* A damper used to block combustion turbine exhaust gases from entering the HRSG.

Damper, Stack.* A damper used to block gas flow through the stack.

Damper, Tight Shutoff. A close-fitting damper to inhibit leakage of air or flue gas into any system component.

Directional Blocking. An interlock that, upon detection of a significant error in HRSG process variables, acts to inhibit the movement of all appropriate final control elements in the direction that increases the error.

Drip Leg. A chamber of ample volume with suitable clean-out and drain connections over which fuel gas is passed so that liquids and solids are trapped.

Duct Burner. A burner located in a duct ahead of or within an HRSG section.

Explosive Mixture. A flammable or combustible mixture in a confined space.

False Start. A condition where the combustion turbine fails to complete its ignition sequence (failure to start) and can result in unburned fuels entering the HRSG enclosure.

Fan, Seal Air. A fan used to supply sealing air.

Fan Test Block Capability. The point on the head versus the flow characteristics curve at which the fan is selected. This is the calculated operating point associated with the maximum continuous rating of the steam generator furnace plus the head and flow margins.

Flame. The visible or other physical evidence of the chemical process of rapid conversion of fuel and air into products of combustion.

Flame Detector. A device that senses the presence or absence of flame and provides a usable signal.

Flame Detector, Self-Checking. A flame detector that automatically, and at regular intervals, tests the entire sensing and signal processing system of the flame detector. This ensures that the failure of any single component cannot result in a false indication of flame.

Flame Envelope. The confines (not necessarily visible) of an independent process that converts fuel and air into products of combustion.

Fresh Air Mode. The operation of an HRSG with atmospheric instead of combustion turbine exhaust.

Fuel Cutback. An action of the combustion control system to reduce fuel flow.

Fuel Gas. See LP-Gas and Natural Gas.

Fuel, JP4.* A light, volatile fuel with a boiling point between gasoline and light distillate.

Fuel, Kerosene.* A light, highly refined fuel. It is slightly more volatile than No. 2 fuel oil.

Fuel Oil.* Liquid fuels defined as Grades 2, 4, 5, and 6.

Fuel-Rich. Indicates a ratio of air to fuel supplied to a burner that provides less than the minimum excess air needed for optimum combustion of the fuel.

Fuel Trip. The automatic shutoff of a specific fuel as the result of an interlock or operator action.

Gas. See LP-Gas and Natural Gas.

Heat Recovery Steam Generator (HRSG). A heat exchanger that uses a series of heat transfer sections (e.g., superheater, evaporator, and economizer) positioned in the exhaust gas flow of a combustion turbine to recover heat and supply a rated steam flow at a required temperature and pressure.

HRSG. See Heat Recovery Steam Generator.

HRSG Control System. The group of control systems that regulates the HRSG process, including the combustion control but not the burner management.

HRSG Enclosure. All ductwork from the combustion turbine exhaust through the steam generator to the stack, including any bypass duct connection.

HRSG Purge. See Purge, Combustion Turbine; Purge, Duct Burner; and Purge Rate.

HRSG System. The unit assembly from the combustion turbine inlet to the flue gas outlet to the atmosphere.

Igniter. A permanently installed device, removable for maintenance only, that provides proven ignition energy for light-off of the main burner.

Igniter, Class 1. An igniter applied to ignite the fuel input through the burner and to support ignition under any burner light-off or operating conditions. Its location and capacity are such that it provides sufficient ignition energy (generally in excess of 10 percent of the full-load burner fuel input) at its associated burner to raise any credible combination of burner inputs of both fuel and air above the minimum ignition temperature.

Igniter, Class 2. An igniter applied to ignite the fuel input through the burner under prescribed light-off conditions. The

range of capacity of such igniters generally is 4 percent to 10 percent of the full-load burner fuel input.

Igniter, Class 3. A small igniter applied, in particular, to gas and oil burners to ignite the fuel input to the burner under prescribed light-off conditions. The capacity of such igniters generally does not exceed 4 percent of the full-load burner fuel input.

Igniter, Class 3 Special. A special Class 3 high-energy electrical igniter capable of directly igniting the main burner fuel.

Initial Operation. The first coordinated operation of the combustion turbine and HRSG.

Interlock. A device or group of devices arranged to sense a limit or off-limit condition or improper sequence of events and to shut down the related equipment or to prevent proceeding in an improper sequence in order to avoid a hazardous condition.

JP4. See Fuel, JP4.

Kerosene. See Fuel, Kerosene.

Listed.* Equipment, materials, or services included in a list published by an organization that is acceptable to the authority having jurisdiction and concerned with evaluation of products or services, that maintains periodic inspection of production of listed equipment or materials or periodic evaluation of services, and whose listing states that either the equipment, material, or service meets identified standards or has been tested and found suitable for a specified purpose.

Logic System. The decision-making and translation elements of the burner management system. A logic system provides outputs in a particular sequence in response to external inputs and internal logic.

- (a) *Hardwired Systems.* Individual devices and interconnecting wiring
- (b) *Microprocessor-Based Systems.*
 1. Computer hardware, power supplies, input/output devices, and the interconnections between these elements
 2. Operating system and logic software

Low Water Cutout. A device arranged to effect a master fuel trip when the water level in the steam drum(s) falls to a predetermined low level.

LP-Gas. A liquefied gas composed of any of the following hydrocarbons or their mixtures: propane, propylene, normal butane, isobutane, and butylenes.

Master Fuel Trip. An event resulting in the rapid shutoff of all fuel to the duct burners, including igniters.

Master Fuel Trip Relay. An electromechanical relay(s) utilized to trip all required equipment simultaneously.

Monitor. To sense and indicate a condition without initiating automatic corrective action.

Natural Gas. A gaseous fuel consisting mostly of a mixture of organic compounds (normally methane, ethane, propane, and butane). The Btu value of natural gases varies from 700 Btu/ft³ to 1500 Btu/ft³ (26.1 MJ/m³ to 55.9 MJ/m³) with the majority averaging 1000 Btu/ft³ (37.3 MJ/m³).

Oil. See Fuel Oil.

Open Flow Path. A continuous path for movement of an airstream through the exhaust to the stack.

Operating Range, Burner. The range between the maximum fuel input and the minimum fuel input within which the burner flame can be maintained in a continuous and stable manner.

Outlet Draft. The flue gas pressure at the outlet of the last convection pass of the steam generator.

Partial Loss of Flame. Loss of flame at any of the separate flame envelopes or burners while flame is maintained at any of the other flame envelopes or burners.

Positive Means of HRSG Isolation. A device in the gas path to the HRSG that will completely prevent turbine exhaust gas entrance to the HRSG and allow personnel entry into the HRSG enclosure (e.g., a blocking plate).

Prove. To establish by measurement or test the existence of a specified condition, such as flame, level, flow, pressure, or position.

Purge, Combustion Turbine. A flow of air at purge rate through the combustion turbine and the appropriate portion of the HRSG enclosure for a sufficient number of volume changes that effectively removes any gaseous or suspended combustibles and replaces them with the purging medium.

Purge, Duct Burner. A flow of combustion turbine exhaust gas or air at purge rate through the HRSG enclosure for a sufficient number of volume changes that effectively removes any gaseous or suspended combustibles and replaces them with the purging medium.

Purge Rate. A constant flow of purging medium at sufficient velocity to achieve a purge.

Reliability. The probability that a generating unit or system will perform for at least a given period of time where used under specified conditions.

Remote Operation. Control from a location that is removed from the combustion turbine and HRSG.

Repeatability. The ability of a device to maintain a constant set point characteristic.

Runback, Combustion Turbine. The controlled unloading of a combustion turbine to a level required by HRSG or other equipment control demands.

Scavenging. The procedure by which liquid fuel left in a burner or igniter after a shutdown is cleared by admitting steam or air through the burner passages, typically through a dedicated scavenging medium valve.

Selective Catalytic Reduction (SCR). A method of reducing NO_x in flue gas.

Separator, Filter, Scrubber. A device incorporated in the main fuel supply line that effectively separates foreign liquids or solids from the fuel.

Set Point. A predetermined value to which a device or system is adjusted and at which it shall be required to perform its intended function.

Shall. Indicates a mandatory requirement.

Should. Indicates a recommendation or that which is advised but not required.

Shutdown, Controlled. The normal unloading and shutdown sequence of plant components based on the manufacturer's criteria.

Shutdown, Emergency. An event resulting in the rapid shutoff of all fuel to the combustion turbine along with a master fuel trip.

Stable Flame. A flame envelope that retains its continuity throughout the maximum rate of change within the operating range of the HRSG.

Stack, Bypass. A stack applied in addition to and separate from the normal HRSG exhaust stack that allows combustion turbine exhaust gas to flow independently to the atmosphere.

Standard. A document, the main text of which contains only mandatory provisions using the word “shall” to indicate requirements and which is in a form generally suitable for mandatory reference by another standard or code or for adoption into law. Nonmandatory provisions shall be located in an appendix, footnote, or fine-print note and are not to be considered a part of the requirements of a standard.

Start-Up Combustion Control System. A control system used to regulate and maintain proper air/fuel ratio during the start-up period where the customary indexes, such as pressure, temperature, load, or flow, that motivate the normal automatic combustion control system are not available or suitable.

Steam Generator. A closed vessel in which water is converted to steam or steam is superheated or in which any combination thereof takes place.

Steam Generator Enclosure. The physical boundary for all steam generator pressure parts and the combustion process.

Supervise. To sense a condition needing attention and automatically initiate corrective action.

Supplementary Firing. The provision of duct burners in an HRSG to increase the temperature of the combustion turbine exhaust gases.

Unit. The confined spaces of the combustion turbine, the HRSG, and the associated ducts that convey the air and combustion gases from the air intake to the stack outlet.

Valve, Charging. A small valve bypassing the main safety shutoff valve that is used for charging the fuel headers and piping and for testing for leaks.

Valve, Check. A valve used to prevent reverse flow.

Valve, Flow Control. A valve capable of regulating quantity of throughput to a controlled range.

Valve, Safety Shutoff. A fast-closing valve that automatically and completely shuts off the fuel supply to main burners or igniters in response to a fuel trip.

Valve, Supervisory Shutoff. A manually operated shutoff valve with a means to provide a “valve closed” position signal.

Valve, Vent. A valve used to allow venting of air or gas from the system to the atmosphere.

Watch-Dog Timer, External. A timer external to a microprocessor-based control that is used to compare the microprocessor cycle timing against itself and that fails safely if the microprocessor timing stops or exceeds the watch-dog time interval.

the owner or the owner’s designated representative from system inception through commercial operation to enhance equipment reliability and personnel safety.

CAUTION

An HRSG is a complex system, often involving numerous components; multiple steam pressure levels; emission control systems; and auxiliary, augmented air, or supplementary firing.

4-1.1.1 The ability of the combustion turbine to satisfy the flow requirements through the HRSG enclosure as required in Section 7-4 shall be considered in the basic design phase.

4-1.1.2 An HRSG system shall be designed to meet the user’s specified modes of operation.

4-1.1.3 System components and control loops shall be compatible and capable of stable operation and control during both steady-state and transient conditions.

4-1.1.4 Control and protective systems and operating sequences shall prohibit the operators from bypassing interlocks.

Exception: As permitted by 2-4.1.3.

4-1.1.5 Training shall ensure operator understanding of the relationships between components; the proper start-up, operation, and shutdown procedures; and the significance of alarms and proper action in response to those alarms.

4-1.2 Explosions and fires have occurred in fuel-fired steam-generating systems as a result of incomplete functional designs. Investigations have revealed human error but often have overlooked the chain of events contributing to the operating error. Therefore, the design, installation, and functional objectives of the overall system components and their controls shall be integrated. Ergonomics and an environment conducive to efficient operator actions and decisions shall be provided.

4-1.3 The planning and engineering phases of plant design and construction shall be coordinated with operating personnel to ensure that operations important to safety can be achieved by qualified operators. Where this is not possible, the needs of operating personnel shall be considered and anticipated.

4-1.4 The proper operator functions, maintenance activities, and training shall be the responsibility of the operating company.

4-2 Project Inception. In the project inception phase, the following shall be accomplished to ensure a plant design that meets expected operating modes and reliability needs:

- (a) Establishment of plant operating parameters
- (b) Identification of site-related constraints
- (c) Review of steam cycle, including generating a family of heat balance diagrams for the expected operating ranges and modes
- (d) Conceptualization of plant layout to provide for personnel safety, operability, and maintenance needs
- (e) Definition and verification of requirements of worst case operating transients, including start-ups
- (f) Definition of required test program
- (g) Definition of start-up criteria and goals
- (h) Identification of the authority having jurisdiction. If multiple authorities having jurisdiction are identified, the

Chapter 4 Project Coordination

4-1 General.

4-1.1 Project coordination, including proper integration of the various system components, shall be the responsibility of

scope of each authority having jurisdiction shall be defined.

4-3 Design.

4-3.1* The project approach shall include full evaluation of all systems and components to ensure compatibility, interface requirements, system dynamics, and the ability to meet all plant operating parameters.

CAUTION

Safety in any plant is the direct result of an extensive up-front effort in the engineering, design, and selection of equipment for each individual application.

4-3.2* This evaluation shall consider the use of dynamic simulation or prior operating experience, or both, before equipment is selected.

4-3.3 Electrical area classifications shall be established by the owner or the owner's designated representative in conjunction with the HRSG system designer prior to commencing detailed design. (*See 5-6.4.*)

4-4 Construction and Installation. The constructor and owner/operator responsible for the erection and installation of the equipment shall verify that all equipment is properly installed and connected.

4-5 Initial Training. The training program shall be specific to the equipment being installed, and all applicable design data shall be made available.

4-5.1 Operator Training. See also 8-2.1.

4-5.1.1 The owner or designated representative shall establish minimum levels of education and competence to be used in the selection and qualification of operators.

4-5.1.2 Prior to commissioning, a formal training program shall be established to prepare personnel to operate equipment safely and effectively.

4-5.1.3 Operator training shall include the following:

- (a) Start-up procedures
- (b) Normal operating procedures
- (c) Normal shutdown procedures
- (d) Emergency shutdown procedures
- (e) Control and safety system check-out and test procedures

4-5.1.4 Where different modes of operation are possible, the procedures identified in 4-5.1.3 shall be prepared for each operating mode. Procedures also shall be prepared for switching from one mode to another.

4-5.2 Maintenance Training. See also 8-2.2.

4-5.2.1 A formal maintenance training program shall prepare personnel to perform any required maintenance tasks safely and effectively. This program shall consist of study or review of maintenance manuals, videotapes, and programmed instruction and testing; field training; and training by equipment manufacturers, among others.

4-5.2.2 Maintenance procedures shall be established to cover routine and special techniques. Any potential safety factors, such as temperature, dust, contaminated or oxygen-deficient atmosphere, internal pressures, and limited-access or confined space conditions, shall be included.

4-5.2.3 Procedures shall be consistent with safety requirements and the manufacturer's recommendations.

4-6 Commissioning.

4-6.1 The HRSG system shall not be operated until safeguards have been tested and demonstrated to operate properly as a system. Any temporary procedures, interlocks, and instrumentation shall be reviewed by the owner or owner's agent, the engineering consultant, the equipment manufacturer, and the operating company, and agreement shall be reached on their suitability in advance of start-up. All temporary modifications shall be documented and permanent resolutions shall be accomplished prior to commercial operation.

4-6.2 The design and function of all safety interlock systems and protective devices shall be reviewed by the organization having HRSG system design responsibility together with those who are to operate and maintain the systems and devices. After installation, coordinated tests of all systems shall be accomplished before initial operation.

4-6.3 The HRSG system shall not be released for commercial operation before the installation and check of the required safeguards and instrumentation system have been confirmed by those responsible for the design and commercial operation.

4-6.4 Documentation of the plant equipment, the system, and maintenance activities shall be updated to reflect accurately changes in the status of equipment and operating procedures.

Chapter 5 Equipment

5-1 General. Equipment required by this standard for safe operation of the HRSG system shall be approved (*see definition of Approved in Section 3-1*) or shall have a demonstrated history of satisfactory and reliable operation for the intended service.

5-2 Combustion Turbine.

5-2.1 Fuel Supply.

5-2.1.1 For fuel oil, two stop valves or equivalent valves in series, with proof of closure, shall be provided in the oil line to the combustion turbine. Means shall be provided to prevent or relieve excess pressure between these valves.

5-2.1.2 For fuel gas, two stop valves or equivalent valves in series, with proof of closure, shall be provided in the gas line to the combustion turbine. An automatic vent valve shall be provided between the two valves.

5-2.2 Purge.

5-2.2.1 The combustion turbine shall have means for purging in accordance with Section 7-4.

Exception: In the event the combustion turbine cannot provide purge in accordance with Section 7-4, other means shall be provided.

5-2.2.2 During the purge sequence, means shall be provided to ensure that fuel does not enter the combustion turbine system downstream of the combustion turbine fuel stop valves.

5-2.2.3 The volume between the combustion turbine stop valves shall be minimized.

5-2.3 Interlocks.

5-2.3.1 Interlocks shall be provided to prevent starting the combustion turbine unless HRSG safety conditions are satisfied. Additional interlocks might be required for unusual intended plant operating scenarios. Typical permissives that shall be considered include the following:

- (a) Water in drum(s) within defined start-up range
- (b) Feedwater supply system available to respond to demand
- (c) Pressure in steam or water spaces not high
- (d) Exit temperature of duct burner(s) not high
- (e) Position of stack closure (if provided) correct
- (f) Pressure in duct system not high
- (g) Augmented air supply in operation (*see* 7-4.2.4)

5-2.3.2 Signals shall be provided to the combustion turbine control system to initiate a change in the combustion turbine operating mode if HRSG safety conditions deviate beyond preset acceptable safety limits. Typical HRSG conditions that warrant a combustion turbine operational response and their corresponding responses shall include the following as listed in Table 5-2.3.2:

Table 5-2.3.2 Typical HRSG Interlock Conditions and Responses

Condition	Response
Water in drum(s) below minimum permitted level	Reduce load*
Gas pressure in combustion turbine exhaust plenum high	Trip combustion turbine
Position of stack closure (if provided) not correct	Trip combustion turbine

*Some steam drum and generating bank designs require a combustion turbine trip if the water level falls below specified levels.

CAUTION

A combustion turbine trip is very costly in terms of its effect on combustion turbine life expectancy and shall be performed only under severe conditions that can result in a safety hazard or significant equipment damage.

5-2.4 Operating Interfaces.

5-2.4.1 The necessity for gradual loading and low-load hold/soak periods for the HRSG shall be evaluated for the following parameters:

- (a) Tube and drum metal temperatures
- (b) Differential metal temperatures within a particular component (i.e., steam drum)
- (c) Rate of change of critical temperatures
- (d) Drum water level

5-2.4.2 In the event HRSG system conditions deviate beyond alarm set points to the interlock limit, the control system shall alarm the condition and initiate a duct burner trip as well as a combustion turbine runback to reduce thermal energy input to the HRSG to a safe level.

5-2.4.3 The control system or the operator shall trip the combustion turbine in the case of an emergency that would lead to a safety hazard or catastrophic failure after the actions of 5-2.4.2 have been accomplished. (*See also* 5-2.3.2.)

5-2.4.4 Consideration shall be given to taking pre-emergency action automatically in the event tripping parameters deviate beyond alarm levels in order to minimize thermal stress duty cycles on the combustion turbine. Runback parameters shall be permitted to initiate from the HRSG or other plant subsystems.

5-3* HRSG Fuel-Burning System.

5-3.1 General.

5-3.1.1 The fuel-burning system shall contain the following subsystems, as applicable:

- (a) Fuel supply
- (b) Main burner
- (c) Igniter
- (d) Atomizing media supply (if included)
- (e) Combustion products removal
- (f) Augmented air supply

5-3.1.2 The fuel-burning system shall provide means for proper start-up, operation, and shutdown of the combustion process. This shall include appropriate openings and configurations in the component assemblies to allow suitable observation, measurement, and control of the combustion process.

5-3.1.3 Each igniter/burner element shall have a purged and cooled flame scanner port embodied in the design.

5-3.1.4 An observation port(s) shall be provided and conveniently located to allow visual inspection of the igniter and main burner flames.

5-3.2 Fuel Supply.

5-3.2.1 General.

5-3.2.1.1 The fuel supply equipment shall be sized and arranged to ensure a continuous fuel flow adequate for all operating requirements of the unit. This shall include coordination of the main fuel control valve, burner safety shutoff valves, and associated piping volume to ensure against fuel pressure transients that might result in exceeding burner limits for stable flame as a result of placing burners in service or taking them out of service.

5-3.2.1.2 The fuel supply equipment shall be designed to inhibit contamination of the fuel. Convenient access to important fuel system components shall be provided. Drains shall be provided at low points in the piping.

5-3.2.1.3 The fuel supply equipment shall be capable of continuing the proper fuel flow during anticipated exhaust gas pressure pulsations at the burner.

5-3.2.1.4 The fuel supply equipment shall be designed with careful consideration of operating environment and ambient conditions, including severe external conditions such as fire or mechanical damage.

5-3.2.1.5 The requirements of the design shall facilitate good housekeeping practices.

5-3.2.1.6 Particular attention shall be given to the integrity of flexible hose or swivel joints.

5-3.2.1.7 The fuel piping materials and system design shall be in accordance with ASME B31.1, *Power Piping*.

5-3.2.1.8* As much of the fuel supply subsystem as practical shall be located away from the burner front. A manual emergency shutoff valve shall be provided that is accessible in the event of fire in the HRSG area.

5-3.2.2 Additional Requirements for Fuel Gas.

5-3.2.2.1 The portion of the fuel supply system that is upstream of the fuel control valve shall be arranged to prevent excessive fuel gas pressure in the fuel-burning system, even in the event of failure of the main supply constant fuel pressure regulator(s). Where full relieving capacity, vented to a safe location, is not installed, the piping system shall be designed for the full supply pressure up to and including all individual burner and igniter safety shutoff valves.

5-3.2.2.2* Positive means to prevent leakage of fuel gas into an idle HRSG shall be provided. An atmospheric vent shall be installed between redundant shutoff valves in any header for main gas or igniter fuel supply.

5-3.2.2.3 Provisions shall be made in the gas piping to allow testing for leakage and subsequent repair. This shall include providing a permanent and ready means for making easy, accurate, periodic tightness tests of the main safety shutoff valves and individual burner safety shutoff valves.

5-3.2.2.4 The discharge from atmospheric vents shall be located so that there is no possibility of the discharged gas being drawn into the combustion turbine air intake, augmented air supply (where provided), ventilating system, or windows of adjacent buildings and shall be extended sufficiently above the HRSG and adjacent structures so that gaseous discharge does not present a fire hazard.

5-3.2.2.5* Manifolding of all vents shall be permitted.

Exception No. 1: Burner vents shall not be manifolded with igniter vents.

Exception No. 2: Header vents shall be manifolded only with other header vents.

Exception No. 3: Vents of headers being served from different pressure reducing stations shall not be manifolded.

Exception No. 4: Vent systems of different HRSGs shall not be manifolded.

Exception No. 5: Vents of systems operating at different pressures shall not be manifolded.

Exception No. 6: Vents of systems using different fuel sources shall not be manifolded.

5-3.2.2.6 All burner safety shutoff valves shall be located as close to the burner as practicable to minimize the volume of fuel left in the burner lines located downstream of the valves.

5-3.2.3 Additional Requirements for Fuel Oil.

5-3.2.3.1 Fill and recirculation lines connected to storage tanks shall discharge below the liquid level to avoid free fall, which might generate static electrical charges as well as increase vaporization.

5-3.2.3.2* Adequate strainers, filters, traps, sumps, and other such items shall be provided to remove harmful contaminants where practical; materials not removed shall be accommodated by special operating and maintenance procedures.

5-3.2.3.3 Special attention shall be given to the routes of piping, valve locations, and other such components in order to minimize exposure to high-temperature or low-temperature sources. Low temperatures might increase viscosity, inhibit flow, or precipitate waxy materials. High temperatures might cause carbonization or excessive pressures and leakage due to fluid expansion in "trapped" sections of the system.

5-3.2.3.4 Means shall be provided to prevent or relieve excess pressure from expansion of entrapped oil in the fuel system.

5-3.2.3.5 Relief valve discharge passages, vents, and tell-tales shall be provided with suitable piping to allow safe discharge of oil or vapors. This piping might need to be heat traced.

5-3.2.3.6 All instruments and control piping and other small lines containing oil shall be rugged, capable of withstanding the expected range of external temperatures, suitably protected against damage, and maintained at the proper tempera-

ture. Interface fluids or sealing diaphragms shall be used where necessary.

5-3.2.3.7 Positive means to prevent leakage of oil into an idle HRSG shall be provided.

5-3.2.3.8* Provisions shall be made in the oil supply system to allow testing for leakage and subsequent repair.

5-3.2.3.9 Fuel oil shall be delivered to the burners at proper temperature and pressure, as recommended by the burner manufacturer, to ensure that the oil is at the viscosity necessary for proper atomization.

5-3.2.3.10 If it is necessary to heat oil, it shall be accomplished without contamination or coking.

5-3.2.3.11 For heated systems, adequate recirculation provisions shall be incorporated for controlling the viscosity of the oil to the burners for initial light-off and for subsequent operation. These systems shall be designed and operated to prevent excessively hot oil from entering fuel oil pumps and causing them to vapor-bind with subsequent interruption to the fuel oil supply.

5-3.2.3.12 Positive means shall be provided to prevent fuel oil from entering the burner header system through recirculating valves, particularly from the fuel supply system of other equipment.

CAUTION

Check valves used for this function have not proven dependable in heavy oil service.

5-3.2.3.13 Atomizing media, where required, shall be supplied free of contaminants that could cause an interruption of service. In addition, adequate insulation and traps shall be included for steam atomizing to ensure the supply of dry atomizing steam to the burners.

5-3.2.3.14 The atomizing medium shall be provided and maintained at the pressure necessary for proper operation.

5-3.2.3.15 Provisions shall be made to ensure that fuel cannot enter the atomizing medium line at any time.

5-3.2.3.16 The atomizing subsystem shall be designed for convenient cleaning and maintenance.

5-3.2.3.17 All burner safety shutoff valves shall be located as close to the burner as practicable to minimize the volume of oil that might be left downstream of the burner valve in the burner lines or that might flow by gravity into the HRSG during an emergency trip or burner shutdown.

5-3.3 Ignition.

5-3.3.1 The ignition subsystem shall be sized and arranged to ignite the main burner fuel input within the limitation of the igniter classification. Igniters are designated by use as Class 1, Class 2, or Class 3 as defined in Chapter 3. Igniters shall be tested to verify that they meet the requirements of the class specified in the design.

5-3.3.1.1 Class 1 Igniters. Class 1 igniters shall be permitted to operate as Class 2 or Class 3 igniters.

5-3.3.1.2 Class 2 Igniters.

5-3.3.1.2.1 Class 2 igniters shall be permitted to operate as Class 3 igniters.

5-3.3.1.2.2 Where Class 2 igniters are provided, the burner shall be operated under controlled conditions to limit the

potential for abnormal operation, as well as to limit the charge of fuel into the duct in the event that ignition does not occur during light-off.

5-3.3.1.2.3 Class 2 igniters shall not be used to ignite the main fuel under uncontrolled or abnormal conditions.

5-3.3.1.2.4 Class 2 igniters shall not be used to extend the turndown range but shall be permitted to be used to support ignition under low-load or adverse operating conditions.

5-3.3.1.3 Class 3 Igniters.

5-3.3.1.3.1 Where Class 3 igniters are provided, the igniter shall be turned off when the trial for ignition period has expired.

5-3.3.1.3.2 Class 3 igniters shall not be used to support ignition nor shall they be used to extend the burner turndown range.

5-3.3.1.3.3 Class 3 igniters shall not be used unless supervision of the individual main burner flame is provided.

Exception: The Class 3 special igniter shall be permitted to be used without supervision of the individual main burner flame while scavenging the main burner.

5-3.3.2 Permanently installed igniters shall be required, and they shall be supervised individually.

5-3.3.3 The ignition equipment shall be located in an appropriate environment with convenient access for maintenance.

5-3.3.4 All igniter safety shutoff valves shall be located close to igniters to minimize the volume of fuel that is downstream of the valves.

5-3.3.5 Igniter parts exposed to combustion turbine exhaust gas, radiation, or flame shall be designed and fabricated of materials capable of withstanding the operating conditions.

5-3.3.6 Igniters shall be suitably shielded from the effects of the combustion turbine exhaust gas to ensure a stable flame under all operating conditions.

5-3.3.7 Ignition devices shall be removable for maintenance while the HRSG is in service. Precautions shall be taken for personnel protection when removing such parts during operation, as they will be hot and hot gases will exit the opening left by the removed part.

5-3.3.8 Ignition transformers shall be housed in an enclosure complying with the relevant requirements of NFPA 70, *National Electrical Code®*, regarding electrical classification and environment and shall be bolted to the duct burner frame and adjacent to the igniter, thereby minimizing the length of high-voltage cable.

5-3.3.9 The ignition transformer shall not be energized before the HRSG enclosure purge is completed. The ignition transformer shall be deenergized at the end of the igniter trial for ignition period.

5-3.4 Main Burner.

5-3.4.1 General.

5-3.4.1.1 Burner Elements. The burner elements shall be designed for operation with the fuel(s) specified. The burner shall be designed to produce a stable flame for its operating range.

5-3.4.1.2 Burner parts exposed to turbine exhaust gas, radiation, or flame shall be designed and fabricated of materials capable of withstanding the operating conditions.

5-3.4.1.3 Provision shall be made for visual observation of the burner flame, including the ignition zone.

5-3.4.1.4 The burner equipment shall be located with convenient access to the burner components and hardware.

5-3.4.1.5* The main burner subsystem shall be designed so that the burner fuel inputs are supplied to the HRSG continuously and within their stable flame limits.

5-3.4.1.6* The limits of stable flame for each burner subsystem producing a separate flame envelope shall be determined by tests without the ignition subsystem in service. These tests shall verify that transients generated in the fuel and combustion turbine exhaust gas subsystems or maldistribution of the combustion turbine exhaust gas do not adversely affect the burners in operation. These tests shall include the expected range of available fuels.

Exception: The requirement to perform tests without the ignition subsystem in service shall not apply to burner systems that require the igniter to be in service any time the burner is being operated.

5-3.4.1.7 Where Class 1 or Class 2 igniters are used, the tests in 5-3.4.1.6 also shall be performed with the ignition subsystem in service to verify that the igniters that are furnished meet the requirements of the class specified in the design. Any resulting extended turndown range shall be available only when Class 1 igniters are in service and flame is proven.

5-3.4.2 Additional Burner Requirements for Fuel Oil.

5-3.4.2.1 Provisions shall be made for cleaning of the burner nozzle and tip.

5-3.4.2.2 Provisions shall be included for clearing (scavenging) the passages of an oil burner into the HRSG with that burner's igniter in service. If the igniter is not operational, the burner shall be removed for clearing.

5-3.5* Augmented Air.

5-3.5.1 Where provided, the fan(s) supplying augmented air to the duct burners shall be operated in accordance with the instructions provided by the supplier of the augmented air system or the organization having responsibility for the overall design.

5-3.5.2 Upon failure of the augmented air supply, means shall be provided to prevent hot gases from exiting the HRSG through the augmented air system.

5-4* HRSG Enclosure.

5-4.1* The HRSG, ducts, and stack shall be sized and arranged to maintain acceptable combustion turbine exhaust gas backpressure and to remove the products of combustion at the same rate that they are generated by the fuel-burning process during operation of the unit.

5-4.2 The HRSG and ducts shall be capable of withstanding a transient pressure determined by the HRSG system designer without permanent deformation due to yield or buckling of any support member.

5-4.3* Proper expansion provision shall be made for the movement of the turbine exhaust duct and the HRSG ducts. Any expansion joints so provided shall withstand at least the highest design pressure for which either duct is designed.

5-4.4* Convenient access and drain openings shall be provided.

5-4.5 The HRSG ducts shall be designed so that they cannot contribute to flame pulsations.

5-4.6 Components common to more than one steam generator shall not limit the rate of removal of products of combustion.

5-4.7 The HRSG ductwork between the combustion turbine outlet and the duct burners shall be designed to provide proper distribution of combustion turbine exhaust gas as required by the burner manufacturer for stable burner operation.

5-4.8 All HRSG units that utilize liquid fuels shall have a duct design that meets the criteria of 5-4.8.1 through 5-4.8.3.

5-4.8.1* All low points shall have sufficient slopes to ensure that no dead pockets exist in the bottom of the ducts at points other than a designed low point.

5-4.8.2* Drains shall be installed, as appropriate, at the low points to facilitate clearing fuel from the HRSG enclosure.

5-4.8.3 Provisions shall be included in the HRSG design and operation to prevent liquid fuels from being absorbed into the insulation that could result in a fire or an explosion.

5-5* Selective Catalytic Reduction.

5-5.1 Where selective catalytic reduction (SCR) systems are selected for NO_x emission control, they shall be integrated into the HRSG design to operate in the flue gas temperature range required.

5-5.2 Areas in which either anhydrous or aqueous ammonia is stored or piped shall be ventilated adequately to preclude toxic or flammable concentrations.

5-5.3 Anhydrous or aqueous ammonia vessels shall be designed to contain their contents at expected elevated ambient temperatures. Overpressure relief valves vented to a safe location shall be provided.

5-5.4 Precautions shall be taken when selecting a storage area for ammonia, as the pressure in storage vessels can rise significantly when exposed to elevated temperatures.

5-6 Electrical.

5-6.1 Electrical equipment shall be protected against transient voltages according to the manufacturer's specification. As a minimum, the system shall function at voltages up to 10 percent above the nominally rated voltage and down to 10 percent below the nominally rated voltage.

5-6.2 All wiring shall comply with the requirements of NFPA 70, *National Electrical Code*.

5-6.3 All high-voltage equipment shall be marked in accordance with the requirements of NFPA 70, *National Electrical Code*.

5-6.4* Where an area is identified as a hazardous location under NFPA 70, *National Electrical Code*, Article 500, the type of equipment enclosure and the wiring methods to be used are specified by that code and shall be followed.

5-6.5 The electrical supply to the burner management system and important subcircuits shall be protected by circuit breakers or fuses.

5-6.6 Upon initiation of a master fuel trip, failure of an electrical power supply shall not impede the orderly and rapid shutdown process.

Chapter 6 Controls, Monitoring, Alarms, and Interlocks

6-1* Control Functions.

6-1.1 General.

6-1.1.1 A single component failure shall not cause loss of the control system's critical functions identified in 6-4.3.

6-1.1.2 Equipment shall be designed and procedures established to permit on-line maintenance of the control equipment. Proper lockout or tagout procedures shall be followed.

6-1.1.3 Procedures for the calibrating and testing of controls and interlocks shall be provided.

6-1.2 Fuel Control.

6-1.2.1 Fuel input shall be controlled to maintain stable firing conditions. Remote manual operation shall be permitted.

6-1.2.2 Minimum and maximum limits on the fuel input shall be established to prevent fuel flow beyond the stable limits of the fuel-burning system.

6-1.3 Feedwater/Drum Level Control. The water level in each drum shall be maintained automatically. Remote manual operation of the feedwater control device shall be available.

6-2 Monitoring.

6-2.1 Information about significant operating events shall be displayed to permit the operator to make a rapid evaluation of the operating situation.

6-2.2 Recording or trend displays of critical parameters taken at intervals no greater than 5 seconds shall be available to the operator at the operator location. (See 6-2.3.)

6-2.2.1 Where accessed through a Cathode Ray Tube (CRT) display in response to an alarm condition, the trend displays shall appear within 5 seconds.

6-2.2.2 Where CRT trend displays are used, the displays shall provide data that is current to within the prior 30 minutes at minimum, and the data provided shall have been stored at intervals of not more than 1 second.

6-2.3 The following HRSG parameters shall be continuously recorded on charts, or the data shall be logged and monitored for trends in accordance with 6-2.2.1 and 6-2.2.2:

- (a) Water level in each steam drum
- (b) Fuel pressure at the duct burner(s)
- (c) Steam pressure at each pressure level
- (d) Duct burner exit temperature before the first tube bank
- (e) Atomizing medium pressure (for oil only)
- (f) Gas temperature upstream of the emissions control catalyst(s)
- (g) HRSG flue gas exit temperature

6-2.4 Consideration shall be given to monitoring the following additional HRSG parameters:

- (a) Fuel flow
- (b) Fuel supply header pressure
- (c) Feedwater pressure at each pressure level
- (d) Feedwater flow at each pressure level
- (e) Economizer inlet water temperature
- (f) Economizer outlet water temperature

- (g) Steam temperature at each level
- (h) Steam flow at each pressure level
- (i) Oxygen in flue gas at HRSG outlet

6-3 Alarms.

6-3.1 Functional Requirements.

6-3.1.1 The functional requirement of any alarm system is to bring a specific abnormal condition to the attention of the operator.

6-3.1.2 Alarms shall be provided to indicate equipment malfunction, hazardous conditions, misoperation, or abnormal conditions that might lead to impending or immediate hazards.

6-3.1.3 The design shall make it difficult to manually defeat the alarm.

6-3.1.4 Where equipment malfunction makes it necessary to defeat an alarm, it shall be done by authorized personnel, and the alarm shall be tagged as inoperative.

6-3.1.5 Alarm systems shall be designed so that, for the alarms required by 6-3.2, the operator receives audible as well as visual signals indicating an abnormal condition. The operator shall be permitted to silence the audible signal.

6-3.2 Required Alarms.

6-3.2.1 General. The alarms indicated in 6-3.2.1.2 through 6-3.2.1.10 shall be required.

6-3.2.1.1 All interlock trips shall be alarmed individually.

6-3.2.1.2 HRSG Steam Pressure (High). High HRSG steam pressure shall be measured at each steam pressure level. It shall warn the operator of a pressure in excess of normal operation.

6-3.2.1.3 Loss of Interlock Power. Loss of interlock power shall be sensed and alarmed and shall include all sources of power required to complete interlock functions. For example, if both a 125-V dc electric circuit and a compressed air circuit are required for an interlock scheme, then loss of either shall be annunciated.

6-3.2.1.4 Loss of Control Power. Loss of control power shall be sensed and alarmed to include any sources of power for the control systems. For example, if both a 125-V dc electric circuit and a compressed air circuit are required for control, then loss of either shall be annunciated.

6-3.2.1.5 Burner Valves Not Closed. The closed position of individual burner safety shutoff valves shall be monitored, and failure of any valve to close following a trip shall be alarmed.

6-3.2.1.6 Drum Water Level (Low). The water level in the steam generator drums shall be monitored and alarmed when the level in any drum drops below the normal operating range.

6-3.2.1.7 Loss of Electrical Generator.

6-3.2.1.8 Duct Firing Temperature (High).

6-3.2.1.9 Scanner Cooling Air Pressure (Low).

6-3.2.1.10 Loss of Augmented Air Supply.

6-3.2.2 Additional Alarms for Fuel Gas.

6-3.2.2.1 Fuel Gas Supply Pressure (High and Low). The gas pressure supplied to the plant shall be monitored at a point as far upstream of the final constant fuel pressure regula-

tor, main fuel control, and main safety shutoff valves as practicable. This is to warn the operator of unusual pressure conditions that might result in damage to equipment or indicate a complete loss of gas supply.

6-3.2.2.2 Fuel Gas Burner Header Pressure (High and Low).

The burner header gas pressure shall be monitored as close to the burners as possible in order to warn the operator of abnormal fuel pressures in advance of duct burner trip conditions.

6-3.2.2.3 Fuel Gas Meter Pressure (High and Low). The pressure at the fuel gas meter shall be monitored at the upstream tap if the fuel gas flowmeter is part of the combustion control system and is not pressure compensated. This shall warn the operator if significant error is present in the flow signal to the control system.

6-3.2.2.4 Ignition Fuel Header Pressure (High and Low).

The ignition fuel header pressure for Class 1 and Class 2 igniters shall be monitored as close to the burners as possible in order to warn the operator of high or low pressure in advance of conditions that lead to a trip.

6-3.2.3 Additional Alarms for Fuel Oil.

6-3.2.3.1 Main Oil Supply Pressure (Low). The oil supply pressure shall be monitored at a point as far upstream of the main fuel control and safety shutoff valves as practicable. This is to warn the operator of unusual pressure conditions that might result in damage to equipment or to indicate a complete loss of supply.

6-3.2.3.2 Fuel Oil Burner Header Pressure (Low). The burner header oil pressure shall be monitored as close to the burners as possible in order to warn the operator of low pressure in advance of conditions that lead to a trip.

6-3.2.3.3 Atomizing Medium Pressure (Low). An alarm shall be provided to warn that the atomizing medium pressure is outside of normal operating range and that poor oil atomization might result.

6-3.2.3.4 Fuel Oil Temperature (Low) or Viscosity (High). For heated fuel oils, an alarm shall be provided to warn that the fuel oil temperature is below, or the viscosity is above, that necessary for proper atomization.

6-4 Interlocks.

6-4.1 Functional Requirements.

6-4.1.1 The HRSG interlock shall protect personnel from injury and shall protect equipment from damage. The interlock system functions shall protect against improper HRSG operation by limiting action to a prescribed operating sequence or by initiating trip devices when approaching an unstable or undesirable operating condition.

CAUTION

It is possible to achieve conditions conducive to an explosion without their detection by any of the mandatory automatic trip devices, even though such devices are properly adjusted and maintained. Therefore, operating personnel shall be made aware of the limitations of the interlock system.

6-4.1.2 Periodic testing and maintenance shall be performed to keep the interlock system functioning properly.

6-4.1.3 Whenever a safety interlock device has been temporarily removed from service for maintenance, testing, or re-

pair, this action shall be noted in the log and annunciated, if practical. Other means shall be substituted to supervise this interlock function.

6-4.1.4 The design of an interlock system shall include the following fundamentals:

- (a) Supervision of the starting procedure and operation to ensure proper operating practices and sequences
- (b) Tripping of the minimum amount of equipment in the proper sequence when the safety of personnel or equipment is jeopardized
- (c) Indication of the initiating cause of the trip and prevention of the start of any portion of the process until proper conditions are established
- (d) Coordination of the necessary trip devices into an integrated system
- (e) Provision of sufficient instrumentation to enable the operator to complete the proper operating sequence in cases where automatic equipment is not available to accomplish the intended function
- (f) Incorporation in the design of as much flexibility with respect to alternate modes of operation as is consistent with good operating practice
- (g) Provision of proper preventive maintenance
- (h) Interlocks that shall not require any deliberate "de-feating" in order to start or operate equipment
- (i) The independence of mandatory master fuel trip sensing elements and circuits from all other control elements and circuits

Exception: Individual burner flame detectors also shall be permitted to be used for initiating duct burner master fuel trip systems.

- (j) Prevention of the misoperation of the interlock system due to an interruption or restoration of the interlock energy supply

6-4.1.5 Interlock functions shall be initiated by one or more of the following:

- (a) Switches independent of control functions and signals
- (b) An analog signal, provided that two analog signals are available with a divergence alarm
- (c) Three analog signals employing an auctioneering system and a divergence alarm or other appropriate fault diagnostic alarm

6-4.2 Flame Detection.

6-4.2.1 Each burner element or zone shall be supervised individually. Upon detection of loss of flame, the associated individual burner safety shutoff valves shall close automatically.

6-4.2.1.1 Where two flame detectors are fitted to each firing element, the flame scanners shall be arranged to alarm on loss of flame from one scanner and to trip the system on loss of flame from two scanners. With one detector out of service, the remaining detector shall trip the system upon loss of flame detection.

6-4.2.1.2 Where Class 1 igniters are provided, the main burner flame shall be proven either by the flame scanner or by proving the igniter. At least one flame detector shall be provided for each burner to detect the burner flame or igniter flame where a Class 1 igniter is provided.

6-4.2.1.3 Burners with Class 2 igniters shall have at least two flame detectors. One detector shall detect the main burner flame and shall not detect the igniter flame. The second detector shall detect the igniter flame during prescribed light-off conditions.

6-4.2.1.4 Burners with Class 3 igniters shall have at least one flame detector. The detector shall detect the igniter flame. It also shall detect the main burner flame after the igniter is removed from service at the end of the main burner trial for ignition.

6-4.2.1.5 Where a self-checking flame scanner is provided to each burner, a burner trip shall occur if the scanner exhibits a self-check fault. Where two self-checking flame scanners are fitted to each burner, the flame scanners shall alarm on loss of flame or the self-check failure of one scanner and shall trip the burner on loss of flame or the self-check failure of two scanners. With one detector out of service, the remaining detector shall trip the burner upon loss of flame or self-check failure.

6-4.2.2 Where a hazardous condition results from loss of flame in more than one burner element or zone, a master fuel trip shall be initiated.

6-4.2.3 It is recognized that any fuel input that does not ignite and burn creates a hazard. Regardless of the number or pattern of flame loss indications used for tripping, flame loss indication on a firing element shall initiate an alarm that warns the operator of a potential hazard.

6-4.2.4 Field testing shall be required to validate basic functions of flame tripping. These tests shall be performed on representative units. The results of these tests might be applied to other units of similar size and arrangements, including firing elements/ nozzles of essentially the same capacity that use similar fuels. These tests shall not be used to replace an acceptance test related to proof of design, function, and components.

6-4.2.5 Flame detector sighting shall be considered in the initial duct burner design. Field tests shall be performed to establish optimum sighting angles of firing elements or igniters and also to check the angular range of the flame detector in relation to the firing elements or igniters.

6-4.3 Duct Burner Master Fuel Trip. A duct burner master fuel trip shall be initiated by the following conditions:

- (a) Low fuel pressure
- (b) Total combustion turbine exhaust flow that drops below the minimum required for safe operation of the duct burner as specified by the burner manufacturer or as proven by trial; it shall be permitted to infer this flow from the operating status of the combustion turbine
- (c) Combustion turbine trip
- (d) Loss of all burner flame
- (e) Partial loss of flame sufficient to introduce a hazardous accumulation of unburned fuel
- (f) Closing of last individual burner safety shutoff valve
- (g) High fuel pressure, if a pressure above the operating limits of the burner(s) can occur
- (h) Low water level on high-pressure section of HRSG
- (i) Loss of energy supply for boiler control, burner management, or interlock system
- (j) Atomizing media supply pressure low

- (k) Burner management system malfunction detected
- (l) Manual trip (*see* 6-5.10)
- (m) Loss of augmented air supply where the safe operation of the duct burner requires augmented air

6-5 Burner Management System Logic. This section provides requirements for the use of logic systems in burner management.

6-5.1 A single failure in the burner management system shall not prevent a required shutdown.

6-5.2 The logic system designer shall recognize the failure modes of components when considering the design application of the system. As a minimum, the following failures shall be evaluated and addressed:

- (a) Interruptions, excursions, dips, recoveries, transients, and partial loss of power
- (b) Memory corruption and losses
- (c) Information transfer corruption and losses
- (d) Inputs and outputs that read “fail on” or “fail off”
- (e) Signals unreadable or not being read
- (f) Addressing errors
- (g) Processor faults
- (h) Common mode failures

6-5.3 Diagnostics shall be included in the design to monitor processor logic function.

6-5.4 Logic failure shall not preclude proper operator intervention.

6-5.5 Logic shall be protected from unauthorized changes. A written procedure shall be provided to control and document authorized upgrades, additions, and deletions.

6-5.6 Logic shall not be changed when the associated equipment is in operation.

6-5.7 System response time (throughput) shall be sufficiently short to prevent negative effects on the application.

6-5.8 Noise immunity shall be adequate to prevent false operation.

6-5.9 No single component failure within the logic system shall prevent a mandatory duct burner fuel trip.

6-5.10 The operator shall be provided with a dedicated manual switch(es) that shall activate the master fuel trip relay independently and directly.

6-5.11 No momentary contact or automatic resetting device, control, or switch that can cause chattering or cycling of the safety shutoff valves shall be installed in the wiring between the load side (terminal) of the primary or programming control and the main or ignition fuel valves.

6-5.12 Requirements for Independence.

6-5.12.1 The logic system performing the safety functions for burner management shall not be combined with any other logic system.

6-5.12.2 This logic system shall be physically separate and visually identifiable.

6-5.12.3 These burner management safety functions shall include but shall not be limited to purge interlocks and timing, mandatory safety shutdowns, trial timing for ignition, and flame monitoring.

6-5.13 Software shall be maintained either in some form of nonvolatile storage or other memory that retains information on the loss of system power.

6-5.14 Application software and input/output devices that support safety logic shall be physically separate from all other software and input/output devices.

6-5.15 System operation shall be verified for compliance with the standard whenever a controller is replaced, repaired, or updated.

6-5.16 Documentation shall be provided to the owner and operator verifying that all safety devices and logic meet the requirements of the application. Functional testing (simulation) of the system shall be performed before it is put into operation.

6-5.17 Programmable logic controllers, if used, shall be monitored by external watch-dog timers. If a timer trips, a duct burner trip shall then occur.

6-6 Operator Interface.

6-6.1 Alarms and indicators shall be grouped operationally and shall be visible to the operator to allow rapid access to operational devices.

6-6.2 All emergency alarm indicators, push buttons, and selector switches shall be readily visible to the operator and shall be labeled clearly. They shall be protected to avoid inadvertent actuation.

6-6.3 All control functions shall be grouped for easy access and in relatively close proximity to their associated alarm and indication devices.

6-6.4* Where CRT displays are used, data shall be displayed on monitor screens in a logical, operational grouping to minimize the number of keystroke operations needed to respond to system upsets. Alarm functions shall be prioritized to appear on the monitor screen upon being sensed regardless of any information already displayed.

6-6.5 Personnel shall be trained to understand and interact with the control systems.

Chapter 7 Purge, Startup, Operation, and Shutdown of HRSG Systems

7-1 General.

7-1.1* This chapter contains requirements for burning fuel gas or fuel oil in duct burners. These requirements shall include certain interlocks for preventing improper action, certain safety trips and flame supervisions, and an indication of the status of the start-up sequence.

7-1.2 This chapter provides minimum standards for design, installation, and operation of duct burners in the exhaust ductwork of a combustion turbine coupled to an HRSG. No specific degree of automation beyond the minimum specified safeguards is defined or shall be required, as this is subject to many factors such as, but not limited to, physical size of the unit, use of the central control room, degree of reliability required, and availability of experienced operating personnel. All devices required in the text shall be provided. The requirements of the operating system that shall be used with these limitations are as follows.

(a) A trained operator shall be available at an appropriate location to take the required safety actions.

(b) The start-up of the burner as a first-time function shall be accomplished by an operator at the burner location who has a direct view of the burner. Recycling of the burner in response to steam demand shall be permitted to be an automatic sequence, provided the combustion turbine has not tripped.

(c) Suitable equipment shall be provided to control HRSG inputs and their rate of relative change within the limits of stable flame throughout the full operating range.

7-2 General Operating Requirements.

7-2.1 Prior to starting a unit, action shall be taken to prevent fuel from entering the HRSG system.

7-2.2 The associated igniter for a burner shall always be used unless the burner is specifically designed to be lit from an adjacent burner. Burners shall not be lit from any hot surface.

7-2.3 Where operating at low capacity, burner fuel pressure shall be maintained above the minimum by reducing the number of burners in service as necessary.

7-2.4 Gas. Before maintenance is performed on the gas header, it shall be purged. (See A-7-1.1.)

7-2.5 Oil. Before maintenance is performed on the oil header, it shall be drained and purged as necessary. (See A-7-1.1.)

7-2.6 Oil — Scavenging of Oil Burner Passages.

7-2.6.1 Oil burner passages shall not be scavenged into a nonoperating HRSG. Combustion turbine exhaust flow shall be functioning and shall be maintained during the scavenging process.

7-2.6.2 Igniters, with ignition established, shall be in service when scavenging oil passages into the HRSG.

7-2.7 Sequencing.

7-2.7.1 Sequencing shall be required to ensure that operating events occur in proper order. Written procedures shall be provided to sequence the start-up and shutdown of the entire unit (combustion turbine and HRSG) properly. Sequencing also shall be utilized when removing burners from operation or adding burners to operation.

7-2.7.2* The start-up and shutdown sequence outlined in this chapter shall be followed.

7-2.7.3* Burners shall not be placed in service or removed from service in a random pattern but shall be placed in service or removed from service in a sequence specified by operating instructions and verified by actual experience with the unit in order to minimize laning or stratification of fuel or combustion products. Burners shall be placed in service as required with fuel flows that ensure proper light-off.

7-2.7.4 If the fuel pressure at the burner header is used as a guide in maintaining the necessary fuel flow per burner, it shall be maintained automatically within prescribed limits as additional burners are placed in service.

7-2.7.5 Duct burners shall be operated in accordance with the manufacturer's specifications and operating procedures.

7-2.7.6 This procedure shall incorporate the following operating objectives:

(a) The purge shall be completed in accordance with Sections 7-4 and 7-5.

(b) No light-off of the duct burner(s) shall occur until after the combustion turbine has established stable operation with an exhaust gas flow not less than that necessary for duct burner operation.

7-2.7.7 Each unit shall be tested during commissioning to determine whether any modifications to the basic procedures are needed to obtain satisfactory ignition and system operation. However, the number of equipment manipulations shall be minimized.

7-2.7.8 The unit shall be operated within the specified parameters. Any modifications or deviations shall be made only after the need for such changes has been determined by operating experience and system review.

7-3 Cold Start Preparation. Preparation for starting shall include a thorough inspection, particularly for the following:

- (a) Unit in good repair and free of foreign material
- (b) Unit inspected for accumulated fuel oil with draining and cleaning performed as necessary
- (c) All personnel evacuated from the unit and associated equipment and all access and inspection doors closed
- (d) All safety shutoff valves proved closed and all ignition sources deenergized
- (e) Fuel gas system vents open and venting to a safe outside location; fuel gas lines drained of condensate
- (f) Circulating valves open to provide and maintain oil flow in the burner headers
- (g) Proper drum water levels established in natural and forced circulation HRSGs and flow established in forced circulation HRSGs
- (h) Burner elements and igniters positioned in accordance with manufacturer's specification
 - (i) Energy supplied to control systems and to safety interlocks
 - (j) Meters or gauges indicating fuel header pressure to the unit
 - (k) Instrumentation tested and functional
 - (l) A complete functional check of the safety interlocks performed after an overhaul or other significant maintenance
- (m) Verification of an open flow path from the inlet of the combustion turbine through the HRSG unit to the stack discharge

7-4 Combustion Turbine Purge and Light-Off.

7-4.1* The purge of the combustion turbine shall be in accordance with the manufacturer's instructions and the requirements of 7-4.2 and 7-4.3. A complete purge of the combustion turbine and applicable portions of the HRSG enclosure shall be accomplished before light-off of the combustion turbine.

7-4.2 Initial Combustion Turbine Purge and Light-Off.

7-4.2.1 Purge prior to the light-off of the combustion turbine shall be accomplished by at least five volume changes and for a duration of not less than 5 minutes. This volume shall be calculated based on the following.

(a) The combustion turbine operating at full load with no supplementary HRSG firing.

(b) The volume from the combustion turbine inlet to the portion of the HRSG where the combustion turbine exhaust gas temperature is reduced to at least 100°F (56°C) below the

lowest autoignition temperature of the fuel(s) for which the system has been designed. However, in no case shall this volume be less than the volume of the HRSG enclosure between the combustion turbine outlet and the outlet of the first evaporator section in the HRSG.

7-4.2.2 The purge rate shall provide sufficient velocity within the connecting duct and HRSG enclosure to ensure sufficient dilution and removal of combustible gases prior to turbine light-off. The adequacy of this purge rate shall be demonstrated by one of the following methods:

- (a) During the purge of the combustion turbine, a flow rate of not less than 8 percent of full-load mass airflow shall be provided through the HRSG, regardless of damper leakage or degradation in the HRSG enclosure.
- (b)* An engineering model of the system from the outlet of the turbine to the outlet of the HRSG shall be created and purge adequacy shall be demonstrated with flow testing performed at the equivalent purge conditions.
- (c) A similar HRSG combustion turbine system burning similar fuel and with similar ductwork and stack geometry has previously been installed and has an extensive documented history of successful start-ups performed in accordance with the manufacturer's recommendations and without occurrences of uncontrolled combustion of fuel accumulations in the HRSG enclosure during turbine light-off.
- (d) Provide combustible gas analyzers that would prevent startup of the combustion turbine if combustible gas concentrations greater than 25 percent of the Lower Explosive Limit (LEL) exist in the HRSG.

7-4.2.3 In the event that the combustion turbine cannot meet the requirements of 7-4.2.2, alternative or supplementary means to satisfy the flow requirements through the HRSG enclosure shall be provided.

7-4.2.4 Where augmented air firing is provided, a means shall be supplied for interlocking and purging the augmented air system prior to initiating and during the combustion turbine purge.

7-4.3 Failure to Start. On failure to start, retrieval of the combustion turbine start shall be permitted following a repurge in accordance with 7-4.2. In the case of liquid fuel, verification also shall be made that the duct low point(s) is cleared of combustibles. (See 5-4.8.)

7-4.3.1 Excessive retries shall be avoided. Where operating experience indicates there are problems in combustion turbine light-off, the light-off attempts shall be terminated and the cause investigated and corrected.

7-4.3.2 Where firing any liquid fuel or any gas that is heavier than air, verification shall be made that the duct low point(s) is cleared of combustibles. This shall be accomplished by one of the following methods:

- (a) For a system firing any liquid fuel, the drains shall be checked to verify that they are clear and that no fuel is present. (See 5-4.8.)
- (b) For a system firing any gas that is heavier than air, the vents shall be checked to verify that no combustible gas is present. (See 5-4.8.)

7-4.3.3 The second trial to start the combustion turbine with the same or alternate fuel shall be permitted following a repurge in accordance with 7-4.2.

7-4.3.4 Subsequent trials to start the combustion turbine with the same or alternate fuel shall be permitted following a repurge in accordance with 7-4.2 and after proving that combustibles have been removed.

7-4.3.5 Stratification of gases shall be considered where analyzing for combustibles.

7-4.4 After completing the purge, the airflow through the combustion turbine shall be permitted to be dropped below the purge rate if required by the design to accomplish combustion turbine ignition.

7-4.5 After successful light-off of the combustion turbine, the combustion turbine shall be brought to speed and loaded as necessary to meet system demands.

CAUTION

The loading of the combustion turbine might be restricted by HRSG parameters.

7-5 Duct Burner Purge and Light-Off.

7-5.1 The duct burner purge shall be accomplished with a flow utilizing air or combustion turbine exhaust at not less than 25 percent of full-load mass flow rate or the minimum flow necessary for operation of the duct burners, whichever is greater.

7-5.2 The duct burner purge shall accomplish at least eight volume changes of the HRSG enclosure after combustion turbine exhaust flow in accordance with 7-5.1 has been achieved.

7-5.3 Purge prior to light-off of the combustion turbine should not be considered a duct burner purge unless the requirements of 7-5.1 and 7-5.2 have been satisfied.

7-5.4 Where augmented air firing is provided, the augmented air plenum and associated ductwork shall be purged into the HRSG enclosure. This requirement is in addition to the requirements of the duct burner purge. (See 7-5.2.)

7-5.5 A duct burner trip or failure to light off duct burners successfully shall require a repurge in accordance with 7-5.1 and 7-5.2 prior to attempting a relight.

7-5.6 Duct Burner. A duct burner purge shall be considered to have been achieved, provided the duct burner purge rate is maintained and all duct burner purge requirements have been satisfied. The duct burner shall be permitted to be lit, or a normal shutdown made, provided that credit for the purge is maintained. Failure of the duct burner purge rate to be maintained or failure to meet any duct burner purge requirement shall require a repurge in accordance with 7-5.1 and 7-5.2.

7-5.6.1 Testing igniters for duct burners shall be conducted in accordance with the following.

(a) A complete, periodic, operational test of each igniter shall be made. The frequency of testing depends on the design and operation history of each individual HRSG and ignition system. As a minimum, the test shall be made during each start-up following an overhaul or other significant maintenance.

(b) Individual igniters or groups of igniters shall be permitted to be tested while the unit is in service. Such tests shall be made with no main fuel present in the igniter's associated burner.

7-5.6.2 Starting Sequence.

7-5.6.2.1* The operating sequences described in 7-5.6.2 assume the use of multiple element duct burners operated inde-

pendently of each other. For installations with a duct burner (single element or multiple element) operated as a single unit, procedures similar to those outlined in 7-5.6.2 shall be followed.

Exception: For duct burners operated as a single unit, sequences of 7-5.6.2 unique to multiple burner operations shall not apply.

7-5.6.2.2* These starting sequences shall be performed in the following order consistent with the manufacturer's instructions.

(a) All duct burner main fuel safety shutoff valves shall be proven to be closed. In addition, all duct burner igniter fuel safety shutoff valves shall be proven to be closed.

(b) The main fuel header and the igniter fuel header shall be pressurized up to the individual main burner and igniter safety shutoff valves in accordance with established operating procedures.

(c) The first individual igniter safety shutoff valve shall be opened and the ignition transformer shall be energized. If flame on the first igniter is not established within 10 seconds, the individual igniter safety shutoff valve shall be closed. The cause of failure to ignite shall be determined and corrected. With turbine exhaust flow maintained, repurge shall not be required, but at least 1 minute shall elapse before attempting a retrial of this or any other igniter.

(d) The main fuel control valve shall be set to the burner light-off position.

(e) The first individual main burner safety shutoff valve shall be opened. If the main burner flame is not proven within 5 seconds after main fuel enters the duct, a duct burner master fuel trip shall occur.

(f) After each stable main burner flame is established, the igniter shall be shut off unless classified as Class 1 or Class 2. The stability of the main burner flame shall be verified.

(g) The associated igniter for a burner always shall be used to light the burner unless the burner is specifically designed to be lit from an adjacent burner. Burners shall not be lit from any hot surface.

(h) Second or succeeding igniters shall be lit in accordance with 7-5.6.2.2(c). If the second or succeeding burner igniter does not light off within 10 seconds after its individual igniter safety shutoff valve has been opened, the individual igniter safety shutoff valve shall close. The cause for failure to light shall be determined and corrected. At least 1 minute shall elapse before the next light-off is attempted.

(i) Second or succeeding burners shall be lit in accordance with 7-5.6.2.2(e). If the second or succeeding main burner flame is not established, the individual burner safety shutoff valve and individual igniter safety shutoff valve shall close. The cause for failure to ignite shall be determined and corrected. At least 1 minute shall elapse before the next light-off is attempted.

(j) After each successive burner light-off, the operator shall verify the flame stability of all operating burners.

7-6 Normal Operation.

7-6.1 The HRSG steaming rate shall be regulated by combustion turbine loading and duct burner exhaust temperature.

7-6.2 The firing rate shall be regulated by varying the fuel to individual burners by means of a fuel control valve(s) or by staged firing where burners are brought in or taken out of service. Individual burner safety shutoff valves shall not be

used to vary the fuel rate of the burner elements. All safety shutoff valves shall be fully open or completely closed; intermediate settings shall not be used.

7-6.3 The burner fuel shall be maintained within a range that falls between the maximum and minimum limits specified by the burner and HRSG manufacturers, or as determined by trial. These trials shall test for minimum load and for stable flame as follows:

- (a) With all burners in service and combustion control on automatic
- (b) With different combinations of burners in service and combustion control on automatic

Where changes occur to the minimum and maximum limits because of various burner combinations and fuel conditions, retesting shall be required.

7-6.4 On loss of an individual burner flame, that burner's individual safety shutoff valve and the associated igniter safety shutoff valve shall close.

7-7 Normal Shutdown.

7-7.1 Burners shall be shut down sequentially as load is reduced by closing the individual burner safety shutoff valves.

7-7.2 The duct burners shall be taken out of service with verification that the safety shutoff valves are secured in the closed position.

7-7.3 When taking the unit (combustion turbine and duct burner) out of service, the combustion turbine load shall be reduced in accordance with the manufacturer's shutdown procedures.

7-7.4 Steam flow shall be maintained through the superheater as necessary.

7-7.5 The normal coast-down of the combustion turbine shall be considered to provide a postpurge of the unit.

7-7.6 After completion of the postpurge in 7-7.5, consideration shall be given to maintaining airflow through the unit in order to prevent accumulation of combustible gases.

7-7.7 Leakage of fuel into the unit shall be prevented.

7-8 Normal Hot Restart.

7-8.1 When restarting a hot combustion turbine, the requirements for cold start preparation as described in Section 7-3(d) through (j) and (m) shall be followed.

7-8.2 The starting sequences of Sections 7-4 and 7-5 shall be followed, if required.

7-9 Duct Burner Emergency Shutdown.

7-9.1 A duct burner master fuel trip shall be initiated by the conditions identified in Chapter 6.

7-9.2 Gas. A duct burner master fuel trip shall stop all fuel flow to the HRSG from all burners by tripping the main and individual burner safety shutoff valves. All vent valves shall be opened. The igniter safety shutoff valve and individual igniter safety shutoff valves shall be tripped and igniter sparks deenergized. Master fuel trips shall operate to stop all fuel flow to the burners within a period of time that does not allow a dangerous accumulation of fuel in the HRSG. All igniters or other ignition sources shall be tripped.

7-9.3 Oil. A duct burner master fuel trip shall stop all fuel flow to the HRSG from all burners by tripping the main and

individual burner safety shutoff valves. The igniter safety shutoff valve and individual igniter safety shutoff valves shall be tripped and igniter sparks deenergized. Master fuel trips shall operate to stop all fuel flow to the burners within a period of time that does not allow a dangerous accumulation of fuel in the HRSG. All igniters or other ignition sources shall be tripped.

7-9.4 The burners shall not be reignited until the initiating trip condition has been investigated and corrected and a duct burner purge has been completed.

Chapter 8 Inspection, Maintenance, Training, and Safety

8-1 Equipment Inspection and Maintenance.

8-1.1 The owner or the owner's representative shall be responsible for ensuring that there is a comprehensive inspection and maintenance program for the equipment. The program shall provide for the maintenance of equipment at intervals consistent with the type of equipment, its service requirements, industry practice, and the manufacturer's recommendations.

8-1.2 Documentation of the plant equipment, the system, and maintenance activities shall be updated to reflect accurately changes in the status of equipment and operating procedures.

8-1.3 As a minimum, the maintenance program shall include the following:

- (a) In-service inspections to identify conditions requiring corrective action or further study
- (b) Detailed, knowledgeable planning for effecting repairs or modifications using qualified personnel, established procedures, and appropriate equipment
- (c) Use of detailed equipment history that records conditions found, maintenance work done, changes made, and date of each
- (d) Written, comprehensive maintenance procedures incorporating manufacturer's instructions that specify tasks and skills required; specification of any special techniques, such as nondestructive testing or tasks needing special tools; and special environmental factors, such as temperature limitations, dust, contaminated or oxygen-deficient atmosphere, internal pressures, and limited access or confined space requirements
- (e) Shutdown maintenance inspections that are thorough in scope, to cover all areas that have been identified during operation as needing attention, in addition to the routine checklist items
- (f) Maintenance of adequate spare parts meeting specifications that provide reliable service without necessitating makeshift repairs

8-1.4 An inspection and maintenance schedule shall be established and followed.

8-1.5 Inspections, adjustments, and repairs shall be performed by trained and qualified personnel, using tools and instruments suitable for the work. Maintenance and repairs shall be accomplished in accordance with the manufacturer's recommendations and applicable standards, codes, and safety regulations.

8-1.6 Operation set points and adjustments shall be verified periodically, and the results shall be documented.

8-1.7 Defects shall be reported and corrected, and the repairs shall be fully documented.

8-1.8 Inspection or maintenance personnel shall not make any changes to the system configuration, including logic, set points, and sensing hardware, without evaluation of the effects on operation and safety and the advance authorization of changes. All changes shall be documented.

8-1.9 In pressurized (positive pressure) installations, inspection or maintenance personnel shall check for and report any combustion-gas leakage through casings, ductwork, expansion joints, or dampers.

8-1.10 Where a unit that fires liquid fuel is out of service and available for inspection, personnel shall check for any accumulation of unburned fuel in the HRSG enclosure, especially in the fin-tube area.

8-1.11 During planned maintenance outages, stack dampers shall be inspected for proper operation and position indication.

8-2 Training.

8-2.1 Operator Training.

8-2.1.1 The owner or the owner's representative shall be responsible for establishing a formal and ongoing program for training operating personnel. The training program shall prepare personnel to operate the equipment safely and effectively. This program shall consist of study or review of operating manuals, videotapes, and programmed instruction and examinations; computer simulation (if available); and supervised hands-on field training. The training program shall apply specifically to the type of equipment and the hazards involved.

8-2.1.2 Before operators shall be permitted to assume their responsibilities, the owner or the owner's representative shall certify that they are trained and competent to operate the equipment under all possible conditions.

8-2.1.3 The owner or the owner's representative shall be responsible for periodic retraining of operators, including review of their competency.

8-2.1.4 The training program and manuals shall be reviewed periodically to keep them current with changes in equipment and operating procedures. The training program and manuals covering operation and maintenance procedures shall be readily available for reference and use at all times.

8-2.1.5 Operating procedures that cover both normal and emergency conditions shall be established. Start-up and shutdown procedures, normal operating conditions, and lockout procedures shall be covered in detail in operating manuals and in the associated training programs.

8-2.1.6 Operating procedures shall be directly applicable to the equipment involved and shall be consistent with safety requirements and the manufacturer's recommendations.

8-2.1.7 Operators shall be trained in the proper procedures for reducing load or tripping the system whenever there is a potential for an unsafe condition that could lead to danger to personnel or property damage. The operator shall be authorized to take appropriate action, including reducing load, tripping equipment, or calling for outside assistance in case of emergency.

8-2.2 Maintenance Training.

8-2.2.1 The owner or the owner's representative shall be responsible for establishing a formal and ongoing program for training maintenance personnel. The training program shall prepare personnel to perform any required maintenance tasks safely and effectively. This program shall consist of study or review of maintenance manuals, videotapes, and programmed instruction and examinations; field training; and training by equipment manufacturers, among others. The training program shall apply specifically to the type of equipment and the hazards involved.

8-2.2.2 Maintenance procedures and their associated training programs shall be established to cover routine and special techniques. Any possible environmental factors such as temperature, dust, contaminated or oxygen-deficient atmosphere, internal pressures, and limited access or confined space requirements shall be included.

8-2.2.3 Maintenance procedures shall be consistent with safety requirements and manufacturer's recommendations. The procedures contained in the training programs shall be reviewed periodically to keep them current with changes in equipment and personnel. They shall be used in the indoctrination and training of new maintenance personnel.

8-2.2.4 Maintenance personnel shall be trained to notify operating personnel in writing of any changes made in safety and control devices.

8-2.2.5 Maintenance personnel shall be trained to be knowledgeable of and to adhere to all Occupational Safety and Health Act (OSHA) safety procedures.

Chapter 9 Combustion Turbine Exhaust Bypass Systems for Unfired HRSGs

9-1* General.

CAUTION

There are several aspects of a gas bypass damper system that raise serious issues of safety and property damage.

9-1.1 The requirements covered in Chapters 1 through 8 shall apply to HRSG configurations unless amended or supplemented in the subsequent sections of this chapter.

9-1.2 The application of dampers, bypass stacks, or other equipment to an HRSG system greatly adds to the complexity of the system. The system designer shall incorporate design features and safeguards for the increased hazards associated with the added equipment.

9-1.3 The principles outlined in this standard shall be incorporated in HRSG configurations that do not fit the designs outlined in Figures A-9-3.1, C-1, and C-2. Additional safety considerations and requirements shall apply where these configurations are used.

9-1.4 The owner or the owner's designated representative shall consider implications of the following points in any application using dampers.

(a) Due to the physical size, shape, and mass, a damper cannot be assumed to reliably fully seal a gas flow path.

(b) Due to a leaking fuel supply system or false start, a possibility exists of having an explosive mixture in the HRSG

enclosure on either side of a gas damper during shutdown periods.

(c) Failure to adequately purge the explosive mixture prior to intentional or unintentional introduction of the hot combustion turbine gas could result in the ignition of the mixture. It is likely that even at full speed and no load, the combustion turbine exhaust temperatures are above the autoignition temperatures for common combustion turbine and HRSG fuels.

(d) Modern combustion turbine exhaust temperatures are sufficient to ignite and sustain the basic iron fire (oxidation reaction).

(e) Systems utilizing two stacks (HRSG and bypass), with or without dampers, can develop a reverse flow of fresh air through the HRSG. This reversal is induced by the high-temperature gas flow up the bypass stack.

(f) Failure of damper operating mechanisms could allow instantaneous reversal of damper position (from open position to closed position and vice versa) due to aerodynamic effects. Sudden closure of an operating combustion turbine free exhaust path will result in extremely high transient duct internal pressures. These high pressures can cause distortion of the HRSG enclosure and rupture of the duct expansion joints.

9-2 Purge.

9-2.1 A purge of both the HRSG enclosure and the bypass system shall be completed as required in Section 7-4 prior to admitting combustion turbine exhaust gas into the HRSG.

9-2.2 Following the purge as required in 9-2.1, it shall be permitted to interrupt the flow through the HRSG using the gas bypass stack. Combustion turbine exhaust flow shall be permitted to reenter the HRSG at a later time without repurging, provided the combustion turbine has been in continuous operation with no trips or misfires.

9-2.3 Where a positive means of HRSG isolation is continuously maintained, the combustion turbine shall be permitted to purge and operate with combustion turbine exhaust through the bypass stack.

Exception No. 1: When the positive means of HRSG isolation is removed, a purge of the HRSG enclosure shall be performed as required by 9-2.1. After this purge, an interruption as allowed by 9-2.2 is again permitted.

Exception No. 2: In the event that the positive means of HRSG isolation is disabled and the combustion turbine can continue to be operated with an exhaust temperature at least 100°F (56°C) lower than the autoignition temperature of fuels designed for use in the specific combustion turbine, the combustion turbine exhaust gas at this lower temperature shall be permitted to be used to purge the HRSG.

9-3 Damper Requirements.

9-3.1* The requirements of Section 9-3 shall apply regardless of physical hardware, that is, single-bladed diverting damper, two separate dampers (single or multi-blade) for isolation or bypass service, or multiple dampers in series with seal air provision.

9-3.2 Damper System.

9-3.2.1 The gas bypass damper failure mode shall be determined, and the exposed system shall be capable of withstanding the resultant transient design pressure as defined in 5-4.2.

9-3.2.2 Where unburned fuels could bypass leaking dampers, a means shall be provided for recognizing leakage of combus-

tion turbine exhaust gas past the closed damper and into the HRSG. Where leakage is detected, the HRSG shall be purged at a temperature at least 100°F (56°C) below the autoignition temperature of the fuel before allowing hot combustion turbine gases to enter the HRSG enclosure. Alternatively, means shall be provided to prevent exhaust leakage into the HRSG enclosure.

9-3.2.3 A positive means of HRSG isolation shall be provided if either of the following operating conditions are encountered:

- (a) Work within the HRSG enclosure is necessary while the combustion turbine is operated through the bypass stack.
- (b) HRSG is empty of water while the combustion turbine is operated through the bypass stack.

Exception: HRSG specifically designed to run dry using special materials, following the guidelines of the HRSG manufacturer.

9-3.2.4 A means of assuring that the HRSG enclosure is free of combustible materials or a means of purging the portion of the HRSG enclosure that is downstream of the positive means of HRSG isolation shall be provided and used to meet the requirements of Section 7-4.

9-3.2.5 Where tight shutoff dampers are utilized, a means to prevent combustible fuel accumulation upstream of the damper shall be provided.

9-3.2.6 During planned maintenance outages, the owner or operator shall be responsible for ensuring that the following is accomplished:

- (a) The damper system inspected for tightness when the damper(s) is fully closed
- (b) The damper operating devices checked for proper operation and positioning
- (c) Correct damper system positioning during purge, start-up, and shutdown verified by functional test

9-3.3 Monitoring.

9-3.3.1 In addition to the requirements in 6-2.3, the position of the bypass stack damper(s) shall be continuously monitored.

9-3.3.2 In addition to the required alarms in 6-3.2, reverse flow through an HRSG (air flow from exhaust stack through HRSG to bypass stack) shall be alarmed if the bypass damper system can be maintained in an intermediate position.

9-3.4 Interlocks.

9-3.4.1 To satisfy the intent of the position of the stack damper starting permissive interlock [see 5-2.3.1(e)], either of the following conditions shall be met:

- (a) The bypass damper proven open
- (b) Both the HRSG isolation damper and the stack damper proven open

9-3.4.2 To satisfy the intent of the position of stack damper interlock (see 5-2.3.2), either of the following conditions shall be met:

- (a) The stack damper proven not to be fully open
- (b) The HRSG isolation damper is proven not to be fully open

Exception: Where the bypass damper is proven fully open.

9-3.4.3 The combustion turbine tripping requirements of 5-2.3.2 shall be followed unless the bypass damper is opened in a manner that prevents unsafe pressurization of the combustion turbine exhaust plenum.

Chapter 10 Fully Fired Systems

(Reserved)

Chapter 11 Referenced Publications

11-1 The following documents or portions thereof are referenced within this standard as mandatory requirements and shall be considered part of the requirements of this standard. The edition indicated for each referenced mandatory document is the current edition as of the date of the NFPA issuance of this standard. Some of these mandatory documents might also be referenced in this standard for specific informational purposes and, therefore, are also listed in Appendix D.

11-1.1 NFPA Publications. National Fire Protection Association, 1 Batterymarch Park, P.O. Box 9101, Quincy, MA 02269-9101.

NFPA 70, *National Electrical Code®*, 1999 edition.

NFPA 8502, *Standard for the Prevention of Furnace Explosions/Implosions in Multiple Burner Boilers*, 1995 edition.

11-1.2 Other Publications.

11-1.2.1 ASME Publication. American Society of Mechanical Engineers, 345 East 47th Street, New York, NY 10017.

ASME B31.1, *Power Piping*, 1995.

Appendix A Explanatory Material

Appendix A is not a part of the requirements of this NFPA document but is included for informational purposes only. This appendix contains explanatory material, numbered to correspond with the applicable text paragraphs.

A-2-1.7 Iron fires in conventional and marine boilers have also occurred principally in air preheaters. At a temperature of approximately 1060°F (571°C), carbon steel fins oxidize at an accelerated rate, and with insufficient cooling, an iron fire is inevitable.

A-3-1 Approved. The National Fire Protection Association does not approve, inspect, or certify any installations, procedures, equipment, or materials; nor does it approve or evaluate testing laboratories. In determining the acceptability of installations, procedures, equipment, or materials, the authority having jurisdiction may base acceptance on compliance with NFPA or other appropriate standards. In the absence of such standards, said authority may require evidence of proper installation, procedure, or use. The authority having jurisdiction may also refer to the listings or labeling practices of an organization that is concerned with product evaluations and is thus in a position to determine compliance with appropriate standards for the current production of listed items.

A-3-1 Authority Having Jurisdiction. The phrase “authority having jurisdiction” is used in NFPA documents in a broad manner, since jurisdictions and approval agencies vary, as do their responsibilities. Where public safety is primary, the authority having jurisdiction may be a federal, state, local, or other regional department or individual such as a fire chief;

fire marshal; chief of a fire prevention bureau, labor department, or health department; building official; electrical inspector; or others having statutory authority. For insurance purposes, an insurance inspection department, rating bureau, or other insurance company representative may be the authority having jurisdiction. In many circumstances, the property owner or his or her designated agent assumes the role of the authority having jurisdiction; at government installations, the commanding officer or departmental official may be the authority having jurisdiction.

A-3-1 Damper, Air Inlet. See DM₄ in Figure A-3-1.

A-3-1 Damper, Bypass. See DM₁ in Figure A-3-1.

A-3-1 Damper, HRSG Isolation. See DM₂ in Figure A-3-1.

A-3-1 Damper, Stack. See DM₃ in Figure A-3-1.

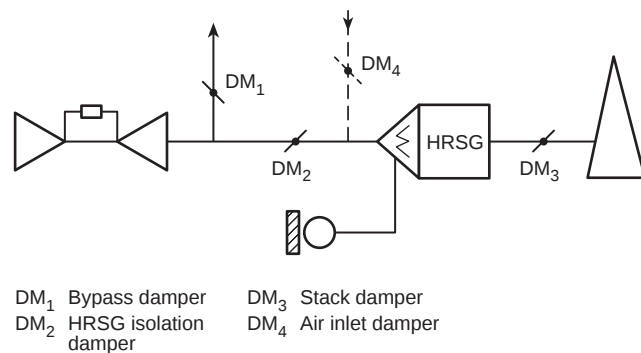


Figure A-3-1 Damper Terminology.

A-3-1 Fuel, JP4. Its properties are defined in MIL-T-5624, *Turbine Fuel, Aviation, Grade JP4, JP5, and JP5/JP8 ST*, and are similar to ASTM D 1655, *Standard Specification for Aviation Turbine Fuels* (Jet B), and ASTM D 2880, *Standard Specification for Gas Turbine Fuel Oils* (OGT).

A-3-1 Fuel, Kerosene. Its properties are defined in ASTM D 396, *Standard Specification for Fuel Oils* (No. 1); ASTM D 1655, *Standard Specification for Aviation Turbine Fuels* (Jet A); or ASTM D 2880, *Standard Specification for Gas Turbine Fuel Oils* (1GT).

A-3-1 Fuel Oil. See ASTM D 396, *Standard Specification for Fuel Oils*, or Grade 2GT in ASTM D 2880, *Standard Specification for Gas Turbine Fuel Oils*.

A-3-1 Listed. The means for identifying listed equipment may vary for each organization concerned with product evaluation; some organizations do not recognize equipment as listed unless it is also labeled. The authority having jurisdiction should utilize the system employed by the listing organization to identify a listed product.

A-4-3.1 The following provides a list of factors that, at minimum, should be considered during the design evaluation:

- (a) Single versus multiple steam pressure levels
- (b) Allowable combustion turbine exhaust backpressure
- (c) Supplementary, auxiliary, or augmented firing
- (d) Combustion turbine exhaust bypass system
- (e) Corrosiveness and fouling of by-products of combustion (for example, fin-tube versus bare-tube, metallurgy, cold-end metal temperature)

- (f) Single or multiple fuels
- (g) SCR or other environmental control systems
- (h) Heat transfer surface cleaning (during operation and shutdowns) and inspection
- (i) Freeze protection
- (j) Rapid start, operating and transients, and thermal shock
- (k) Dry operation
- (l) Protective systems
- (m) Degree of automation and complexity of control systems
- (n) Operator interface
- (o) Overall system performance evaluation, feedback, and iteration (expert system database consideration)
- (p) Description of start-up validation test program (reference test cases and simulator data library where available)
- (q) Combustion turbine (purge exhaust)
- (r) Bypass stack and damper
- (s) HRSG and interconnecting ducts
- (t) Forced draft fan, induced draft fan, or discharge stack, in any combination
- (u) Burner management system logic
- (v) Flame monitoring and tripping systems
- (w) Combustion control system
- (x) Power supplies configuration and codes
- (y) Piping system configuration and codes
- (z) Operating information
- (aa) Input/output selection
- (bb) Information displayed
- (cc) Data transmission (noise accuracy considerations)
- (dd) Programmable logic controller software and hardware considerations
- (ee) Requirements for operation from a remote location
- (ff) Initial control tuning

A-4-3.2 Dynamic simulation, where utilized, should include development of the following:

- (a) Configuration and data initialization
- (b) Plant behavior knowledge
- (c) Preliminary control system design and tuning
- (d) Validation of operating requirements (system performance)
- (e) Transients and ramps for intended and unintended operation

A-5-3 The operation of the HRSG combustion turbine system differs from a conventional multiple burner boiler. Some of the differences include the following:

(a) The combustion turbine is an internal combustion engine. The engine burns a lean mixture of fuel with compressed air. The hot, pressurized combustion gases expand from the combustion chamber through a series of rotating turbine wheel and blade assemblies, resulting in shaft power output and hot turbine exhaust gas discharge to the HRSG. Turbine exhaust gas is hot and has a reduced oxygen content relative to air.

(b) The design of the HRSG differs from that of a regular steam generator in that, in most cases, the HRSG is designed

to utilize the residual heat from the combustion turbine exhaust gas, with some supplementary firing by the duct burner, if necessary.

(c) Because the combustion turbine is a volumetric machine, combustion turbine exhaust gas is discharged within closely prescribed limits, with the oxygen content varying as a function of load.

(d) Separate purge requirements exist prior to combustion turbine light-off and prior to duct burner light-off.

(e) Air/fuel ratios controlled by duct burners are neither possible nor recommended. As vast quantities of turbine exhaust gases that are far in excess of the stoichiometric requirements of the fuel are utilized, fuel-rich conditions cannot inherently occur under normal controlled operating conditions.

(f) Many types of burners are available for HRSG systems. The burner can consist of a number of parallel tubes or runners placed in the duct to provide the required heat release. This commonly is used for gaseous fuels and is referred to as a "grid" burner. Alternatively, wall-mounted burner systems with parallel flame holders within the duct can be used for liquid fuels. In-line register-type burners manufactured in Europe also have been used. Ignition systems for these burner types can employ Class 1, Class 2, or Class 3 igniters.

A-5-3.2.1.8 Protection for burner front exposed equipment might justify installation of fire protection as indicated in NFPA 850, *Recommended Practice for Fire Protection for Electric Generating Plants and High Voltage Direct Current Converter Stations*.

A-5-3.2.2.2 Atmospheric vent valves located between shutoff valves are intended to relieve any gas pressure that builds up due to failure of the first (upstream) shutoff valve. This minimizes the potential for leakage into an idle HRSG. To perform properly, these valves should be large enough to relieve gas to the atmosphere at a rate equal to the potential leakage rate. In the absence of equivalent protection, vent pipe sizes and vent valve port diameters should conform to Table A-5-3.2.2.2. Special precautions should be taken to safely vent heavier-than-air gases.

Table A-5-3.2.2.2 Vent Line Sizes

Gas Supply Line Size		Shutoff System Vent Port Size	
in.	mm	in.	mm
≤1½	≤38	¾	19
2	50.8	1	25.4
2½ to 3	64 to 76	1¼	31.8
3½	89	1½	38
4 to 5	102 to 127	2	50.8
5½ to 6	139.7 to 152.4	2½	64
8	203	3½	89
>8	>203	15% of supply line cross-sectional area	

A-5-3.2.2.5 Where vents are manifolded from safety shutoff systems, the cross-sectional area of the manifold pipe should be equal to, or greater than, the sum of the cross-sectional areas of the two largest vents involved.

A-5-3.2.3.2 Contaminants in fuel oil might include salt, sand, sludge, water, and other abrasive or corrosive constituents.

Some fuel oils contain waxy materials that precipitate out, clogging filters and other elements of the fuel system.

A-5-3.2.3.8 A permanent and ready means for making easy, accurate, periodic tightness tests of the main safety shutoff valves and individual burner safety shutoff valves generally is not feasible for fuel oil systems. However, a visual check of the burners and combustion area prior to starting operation can determine if leakage has occurred. Also, pressurizing the oil supply header with the individual burner safety shutoff valves and the recirculating valves closed indicates if leakage is present. If oil pressure remains within defined limits, it can be concluded that the individual burner safety shutoff valves are not leaking.

A-5-3.4.1.5 Variations in burning characteristics of the fuel and the normal variations in fuel-handling and fuel-burning equipment introduce an uncertainty to the lower operating limits of the main fuel subsystem in any given HRSG design. In these circumstances, Class 1 or Class 2 igniters, as demonstrated by test, can be used to maintain stable flame.

A-5-3.4.1.6 Such transients are generated by burner shutoff valves, dampers, and other components that operate at speeds faster than the response speeds of other components in the system.

A-5-3.5 See Figure A-5-3.5.

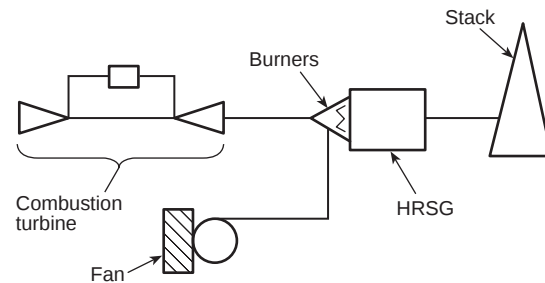


Figure A-5-3.5 HRSG burners with augmented combustion air supply.

A-5-4 Proper design consideration should be given to internal insulation and cover plates so that the following factors are properly addressed:

- Insulation thickness and external casing temperature calculations
- Internal plate thickness and material
- Pin pitch, diameter, and fixing methods
- Welding procedures

A-5-4.1 The external skin temperature and acoustical design of the HRSG, HRSG ducts, and HRSG stack should comply with the requirements of federal, state, and local regulations.

A-5-4.3 Any through-duct penetrations should have proper provision for expansion and sealing. Where pipes, tubes, headers, or drums create a through-duct penetration, calculations should be provided to demonstrate that the differential expansion and the sealing can properly accommodate such expansion.

A-5-4.4 Access should be provided for proper maintenance and repair. This should include personnel access where allowed by ducting. All pressure parts should have access for periodic inspection and for mandatory hydraulic tests.

A-5-4.8.1 Consideration should be given in the design to minimizing pockets that could trap combustible materials.

A-5-4.8.2 Drains should be provided in all ducts or enclosures where fluid accumulation is possible.

A-5-5 Some authorities having jurisdiction require the installation of selective catalytic reduction systems in some HRSG systems to reduce the emissions of NO_x . Because such a system has a narrow range of optimum operating temperatures and is subject to maximum temperature limitations lower than many combustion turbine full-load exhaust temperatures, it usually is installed between heat transfer surfaces within the HRSG.

The chemical process of reduction necessitates the addition of ammonia to reduce NO_x to nitrogen and water in the presence of the catalyst. In addition, if the fuel contains sulfur, a reaction that results in the formation of ammonium bisulfate can occur. This material tends to deposit on both the catalyst and metallic surfaces downstream of the reactor section, primarily at low temperatures. Although troublesome in terms of corrosion, fouling, and material life, ammonium bisulfate does not directly affect flame safety.

It is common practice to use either anhydrous or aqueous ammonia as the reducing agent in a selective catalytic reduction (SCR) system. These chemicals are not interchangeable, and a specific system design is needed, depending on the form to be used at a particular installation. Both forms, on release, are considered a potential health hazard. Ammonia gas is flammable in air at concentrations between 16 percent to 25 percent by volume. Such concentrations usually are not encountered. The system should provide the necessary features to ensure such concentrations cannot occur during abnormal conditions.

Aqueous ammonia usually is stored in a closed vessel to prevent the release of vapor. Such vessels are designed for low [less than a gauge pressure of 50 psi (344.7 kPa)] pressures and only approach the design pressure under high ambient temperature conditions. Due to the corrosive nature of ammonia, material selection is an important consideration.

Anhydrous ammonia is stored in a concentrated liquid-vapor form within closed vessels. Under ambient temperature conditions, higher pressures than those observed with aqueous ammonia can result. Vessels built in accordance with the ASME *Boiler and Pressure Vessel Code* are required with design gauge pressures of 250 psi (1723.7 kPa) or higher. The following sources provide additional information and requirements for storage and handling of anhydrous ammonia: CGA G-2, *Anhydrous Ammonia; Code of Federal Regulations*, Title 29, Part 1910.111, "Storage and Handling of Anhydrous Ammonia"; and ANSI K61.1, *Safety Requirements for the Storage and Handling of Anhydrous Ammonia*.

A-5-6.4 For guidance in determining area classification, see NFPA 497, *Recommended Practice for the Classification of Flammable Liquids, Gases, or Vapors and of Hazardous (Classified) Locations for Electrical Installations in Chemical Process Areas*; NFPA 499, *Recommended Practice for the Classification of Combustible Dusts and of Hazardous (Classified) Locations for Electrical Installations in Chemical Process Areas*; NFPA 497M, *Manual for Classification of Gases, Vapors, and Dusts for Electrical Equipment in Hazardous (Classified) Locations*; NFPA 70, *National Electrical Code*, Article 500; and API RP 500A, *Classification of Locations for Electrical Installations in Petroleum Facilities Classified as Class I, Division 1, and Division 2*.

A-6-1 NOTE: The user of NFPA 8506 is encouraged to use judgment in the application of these guidelines for all process

and safety functions contained in the distributed control system. This section is not intended to apply specifically to burner management systems.

(a) *Data Transmission.*

1. Every input should be sampled at an interval no longer than 1 second. Every output should be updated at an interval no longer than 1 second.
2. For protective actions, the system should be able to convert a changed input sensor value to a completed output control action in less than 250 milliseconds.
3. Changes in displayed data or status should be displayed within 5 seconds.
4. Data acquisition and transmission systems should be protected from noise pickup and electrical interference.
5. In redundant systems, the data links should be protected from common mode failures. Where practical, redundant data links should be routed on separate paths to protect against physical damage that disables both data links.

(b) *Hardware.*

1. The hardware selected should have adequate processor capacity to perform all the functions required for start-up sequencing, normal operation alarming, monitoring, and shutdown of the controlled equipment. Capacity also should be available for data storage and sorting; this can be permitted to be located in a separate processor.
2. Selection should take into consideration the requirements for reliability, maintainability, and electrical classification.
3. The hardware should provide for automatic tracking between auto-manual functions to allow for immediate seamless transfer.
4. The hardware should be capable of stable dynamic control.
5. The hardware should be capable of thorough self-diagnosis.
6. Consideration should be given to all levels and types of electrical interference that can be tolerated by the hardware without compromising its reliability or effectiveness.
7. Fail-safe operation should be obtained through a thorough and complete analysis of each control loop and by providing for a failure of that loop (i.e., valve/actuator) to cause a fail-safe position.

(c) *Software.*

1. The software package should be designed to include all logic to provide a safe and reliable control system. When the software calls for the operation of a field safety device, a feedback signal should be provided to prove that the requested operation has taken place, and an alarm should be actuated if the action is not confirmed in a specified time.
2. The software package should be checked to ensure that no unintended codes or commands are present (e.g., viruses or test breaks). The software package should be tested and exercised before being loaded into the plant site computers or processors.

3. The software system should be protected from inadvertent actions from operators and also should be tamper-proof.
4. Written procedures should specify the functions that can and cannot be accessed by the operator and those functions that require additional authorization for access.
5. The software should be permitted to provide for authorized on-line changes of the timers and set points, provided the safety of the operating equipment is not compromised.
6. The software should implement and enhance the self-diagnostic hardware that has been provided.

A-6-6.4 A sequence of events recorder, where provided, should time-tag events with a resolution of 10 milliseconds or less.

A-7-1.1 Common hazards are involved in the combustion of liquid and gaseous fuels. In addition, each of these fuels has special hazards related to its physical characteristics.

(a) *Gas Firing.* The following items should be considered in the design of the fuel gas firing systems:

1. Gas is colorless; therefore, a leak usually cannot be detected visually. Also, reliance cannot be placed on detection of a gas leak by means of the presence of odor.
2. Potentially hazardous conditions are most likely to occur within buildings, particularly where the gas piping is routed through confined areas. In the latter instance, adequate ventilation should be provided. Outdoor steam generators tend to minimize confined area problems.
3. Natural gas can be either "wet" or "dry." A wet gas usually implies the presence of distillate, which could be characteristic of a particular source. In the case of such a wet gas, the carryover of distillate into the burners could result in a momentary flameout and possible reignition. Reignition could result in an explosion. Therefore, special precautions should be taken with wet gas supply systems. (See NFPA 54, *National Fuel Gas Code*.)
4. Discharges from relief valves or from any other form of atmospheric vents can become hazardous unless special precautions are taken.
5. Maintenance and repair of gas piping can be hazardous unless proper methods are used for purging and recharging the line before and after making the repairs. (See NFPA 54, *National Fuel Gas Code*.)

(b) *Oil Firing.* The following items should be considered in the design of the fuel oil firing systems:

1. Firing of oil fuel into an HRSG can create a special hazard by causing soot accumulations in low-temperature sections.
2. Small oil leaks can result in serious fire damage.
3. Water or sludge in fuel oil storage tanks or improperly located suction takeoffs from the storage tank can result in hazardous interruptions or pulsations of the fuel supply to the burners. A flameout, either immediately or at a later time, can result because of plugged strainers or burner tips.
4. Widely different characteristics of fuel oil from either a single source or multiple sources can result in a

significant change in British thermal unit (Btu) input rate to the burner(s). Different shipments of fuel oil with dissimilar characteristics can cause a precipitation of sludge that can lead to hazards as described in A-7-1.1(b)3.

5. Inserting an oil gun in the burner assembly without a tip, new gaskets, or a sprayer plate is a constant hazard. This can result in an unsafe operating condition.
6. Clear distillate fuels have low rates of conductivity and generate static electrical charges in the fuel stream that can be dangerous unless flowing velocities are limited. (See NFPA 77, *Recommended Practice on Static Electricity*, and API RP 2003, *Protection Against Ignitions Arising Out of Static, Lightning, and Stray Currents*.)
7. Maintenance and repair of oil piping can be hazardous unless proper methods are used for purging and recharging the line before and after making repairs. (See NFPA 31, *Standard for the Installation of Oil-Burning Equipment*.)
8. The incompressibility of fuel oil can create very rapid transients in oil flow through operating burners when the following occur:
 - a. Rapid operation of the oil supply valve
 - b. Rapid operation of individual burner shutoff valves
 - c. Rapid operation of the regulating valve in the return oil line from the burner header (on systems using this type of control)

(c) *Fuel Oil — General Considerations.*

1. The term fuel oil refers to liquid fuels with widely differing characteristics. A fuel oil burning system is designed for a specific range of oil characteristics. Attempting to burn an oil whose characteristics differ widely from those for which the system is designed can cause serious operating difficulties and potential safety hazards. Therefore, care should be exercised to ensure that fuel oil received at a plant is within the specified range of the handling and burning equipment.
2. The more important characteristics of fuel oils are provided in ASTM specifications. It is relatively simple to identify oils that require special provisions for storing and functions such as heating, pumping, and atomizing. Generally speaking, Grades 2 and 4 have lower viscosities and less water and sediment than Grades 5 or 6 and so require fewer special provisions to ensure proper handling and burning. However, most boiler fuel oil systems are designed for Grades 5 and 6, which are heavier; therefore, such systems include provisions for preheating these usually viscous fuels. Furthermore, more care is necessary in the design and operation of fuel oil systems supplied with Grade 6 oil than with the other ASTM grades. Care should be taken to avoid flameouts attributable to interruptions or pulsation of the fuel supply or plugging of strainers or burner tips.
3. All of the following characteristics can have an effect on the proper and safe burning of fuel oils:
 - a. Fuel oil is a complex mixture of hydrocarbons of differing molecular weights and differing boiling and freezing points. When subjected to sufficiently high temperature, accumulations of the fuel par-

tially decompose and volatilize, thus creating new liquid, gaseous, and solid fuels with unpredictable properties.

- b. Fuel oil should be introduced into the furnace as an extremely fine mist in order to mix intimately with the combustion air so that it can burn quickly and completely. In boilers, this is accomplished by spraying the fuel oil through small orifices with high-pressure drops (mechanical atomization) or by using steam or air to break up small oil streams. Viscosity and volatility are characteristics of the oil that indicate ease of atomization.
- c. Viscosity affects ease of pumping and atomization. Temperature significantly affects viscosity.
- d. Flash point is an indicator of the volatility and, thus, of potential for combustible vapors.
- e. Some fuel oils contain constituents that, when overheated, can decompose and form solids or can solidify when exposed to low ambient temperatures. The presence of such solids in the fuel can cause interruptions.

A-7-2.7.2 This sequence provides a continuous airflow or flue gas flow through the HRSG at a rate that is at least the rate that existed during the purge operation. The objective of this practice is to ensure minimum velocities through the unit to prevent hazardous accumulations of unburned fuel.

A-7-2.7.3 Automatic start systems can be permitted to establish igniter flame on multiple burner elements simultaneously with proper supervision. Similarly, main burner elements can be permitted to be configured to operate as one with proper supervision.

A-7-4.1 The objective of this practice is to remove potential accumulations of hazardous unburned fuel from the volume defined in 7-4.2.1 that could be ignited by light-off of the combustion turbine.

A-7-4.2.2(b) An engineered model study has shown the capability of accurately depicting the gas flow characteristics within an HRSG enclosure. The greatest experience base for model studies has been physical scale models utilizing a cold airflow. These models are generally accurate if kinematic similitude is maintained by ensuring that the test flow within the model is fully turbulent (Reynolds Number $>3,200$). The increasing availability of computational fluid dynamics (CFD) programs offers an alternative to physical modeling. With any model study, the accuracy of the results are dependent on the skill of the modeler, the quality of the modeling tools, and the detail of the model.

Engineered model studies can be used to determine purge effectiveness. For this use, the physical model generally offers the advantage of ease of visualization when identifying areas of high combustibles concentrations. It is difficult, however, to quantify purge efficiencies with a physical model. The CFD model, when properly created, is able to define the change in concentration of the combustibles, but in general, is less easily visualized. It is also recognized that it is possible to consolidate the results of many model studies to develop a calculative method of correlating HRSG configurations with specific purge rates.

Physical or computational modeling or other engineering approaches can demonstrate or define a specific purge rate (higher or lower than those specified in 7-4.2.2) for HRSGs.

However, the parameters vary for each installation and are unique.

As a minimum, consideration should be given, but not be limited to, the following:

- (a) Ductwork and stack geometry
- (b) HRSG geometry
- (c) Fuel characteristics (lower explosive limits, autoignition temperature, density, etc.)
- (d) Combustion turbine/ HRSG conditions (hot or cold)
- (e) Fuel accumulation zones

In general, when using physical or computational modeling, the reproduction of areas where gas can accumulate is of extreme importance.

An example of this would be the steam drum and collection header cavities located on the top of HRSGs. These pockets can be very large and vary in size considerably, depending on the manufacturer and model. Low purge rates might not be able to dissipate the flammable vapors in these locations and could require separate venting or blowers to reduce the flammable vapors.

CAUTION

While not covered by this standard at the present time, interconnected multiple combustion turbines and HRSGs are being installed and operated. These configurations require special purge considerations and could require purge rates significantly higher than required by this standard.

Follow the principles provided in NFPA 69, *Standard on Explosion Prevention Systems*, to determine the required purge air flow rate. When applying this method, a k factor consistent with the HRSG design should be established.

A-7-5.6.2.1 Figures A-7-5.6.2.1(a) through (d) represent typical fuel supply systems for duct burners that operate as a single unit.

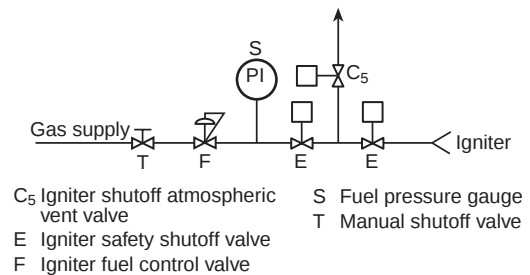


Figure A-7-5.6.2.1(a) Typical duct burner gas ignition system of a single element or multiple elements fired simultaneously (Class 3 igniter monitoring requirements shown).

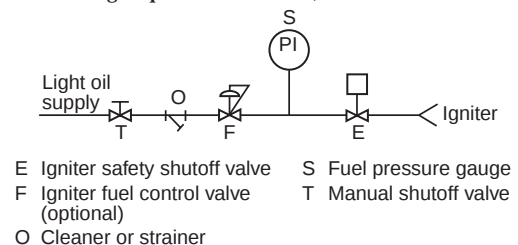


Figure A-7-5.6.2.1(b) Typical duct burner oil ignition system of a single burner or multiple burners fired simultaneously (Class 3 igniter monitoring requirements shown).

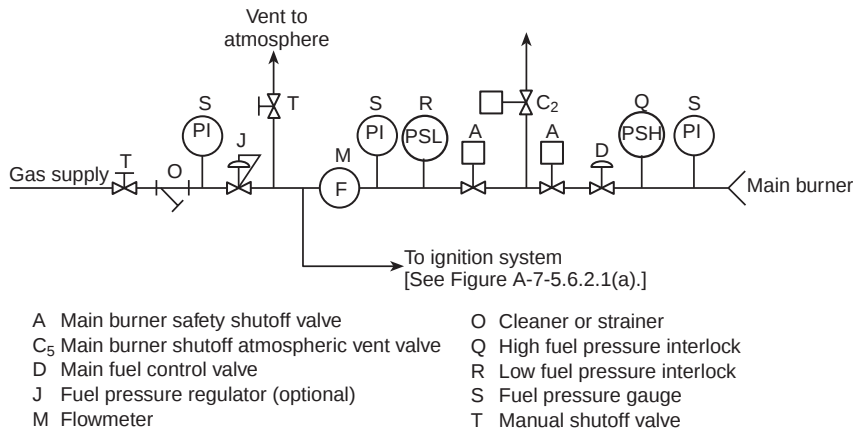
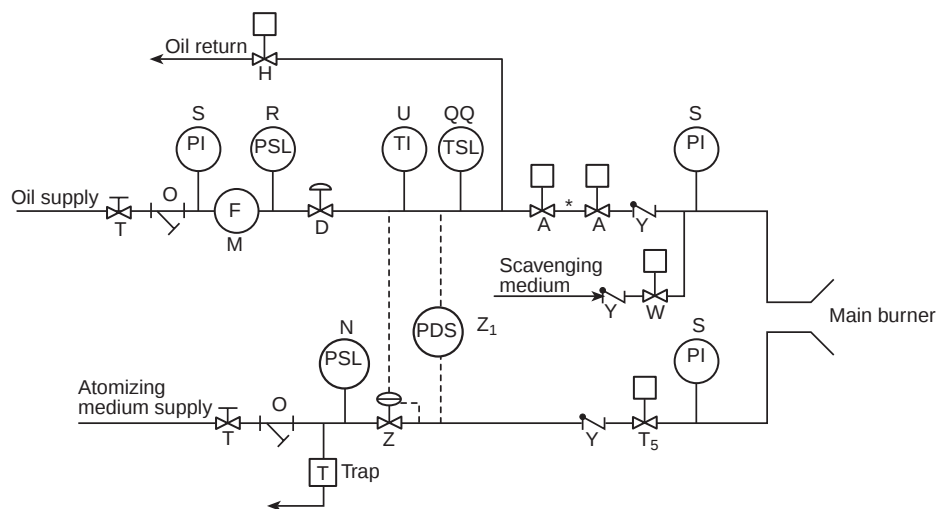


Figure A-7-5.6.2.1(c) Typical main gas duct burner system of a single element or multiple elements fired simultaneously.

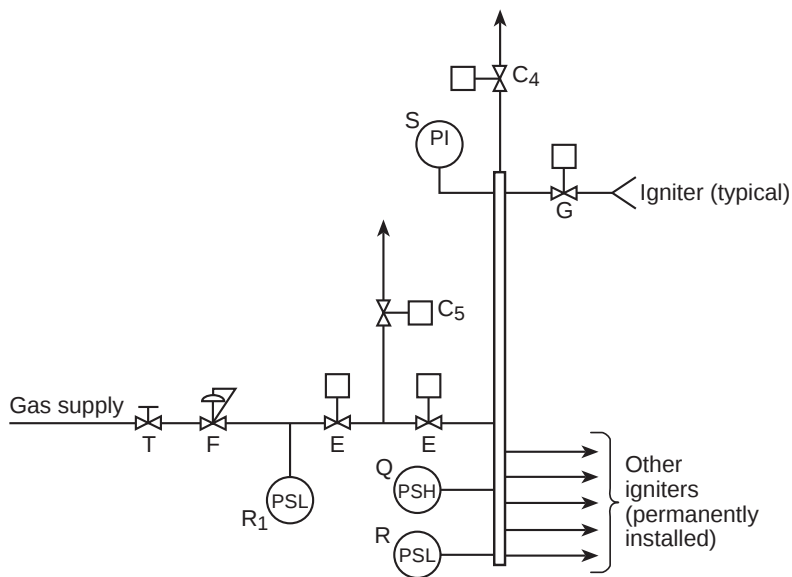


*Caution: See Section 5-3.2.3.4.

- | | |
|--|--|
| A Main burner safety shutoff valve | S Fuel pressure gauge |
| D Main fuel control valve | T Manual shutoff valve |
| H Recirculating valve (optional for unheated oil) | T ₅ Atomizing medium individual burner shutoff valve, automatic |
| M Flowmeter | U Temperature gauge (optional for unheated oil) |
| N Low atomizing pressure interlock | W Scavenging valve |
| O Cleaner or strainer | Y Check valve |
| QQ Low-temperature or high-viscosity alarm (optional for unheated oil) | Z Differential pressure control valve |
| R Low fuel pressure interlock | Z ₁ Differential pressure alarm and trip interlock |

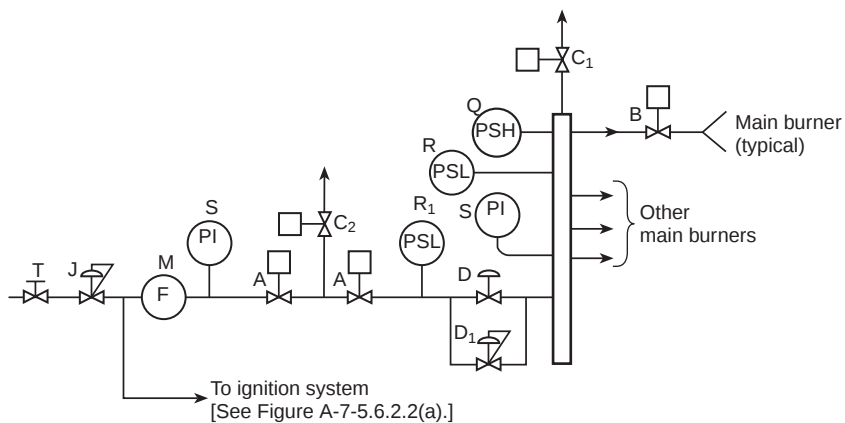
Figure A-7-5.6.2.1(d) Typical steam or air atomizing single main oil duct burner system.

A-7-5.6.2.2 Figures A-7-5.6.2.2(a) through (f) represent typical fuel supply systems for duct burners that operate as a single unit.



- | | |
|---|---|
| C ₄ Igniter header atmospheric vent valve (optional) | Q High fuel pressure interlock |
| C ₅ Igniter supply atmospheric vent valve | R Low fuel pressure interlock |
| E Igniter header safety shutoff valve | R ₁ Low fuel pressure interlock (alternate location for R) |
| F Igniter fuel control valve | S Fuel pressure gauge |
| G Individual igniter safety shutoff valve | S Manual shutoff valve |

Figure A-7-5.6.2.2(a) Typical duct burner gas igniter system.



- | | |
|--|---|
| A Main safety shutoff valve | M Flowmeter |
| B Individual burner safety shutoff valve | Q Burner header high fuel pressure interlock |
| C ₁ Main burner header charging atmospheric vent valve (optional) | R Burner header low fuel pressure interlock |
| C ₂ Main burner header shutoff atmospheric vent valve | R ₁ Burner header low fuel pressure interlock (alternate location for R) |
| D Main fuel control valve | S Fuel pressure gauge |
| D ₁ Main fuel bypass control valve (optional) | T Manual shutoff valve |
| J Fuel pressure regulator (optional) | |

Figure A-7-5.6.2.2(b) Typical main gas duct burner system.